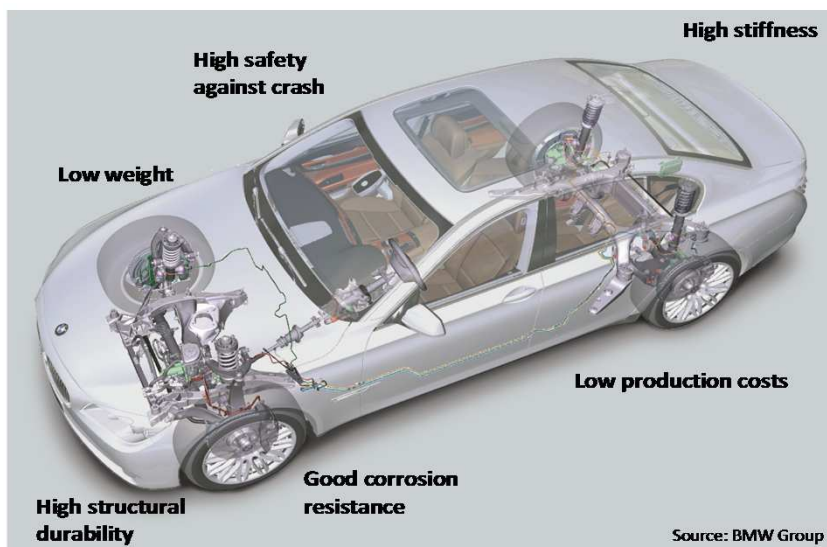


Multiaxial Fatigue of Welded Joints under the Aspect of Local Assessment Concepts

C. M. Sonsino

Chassis of a 7 Series BMW



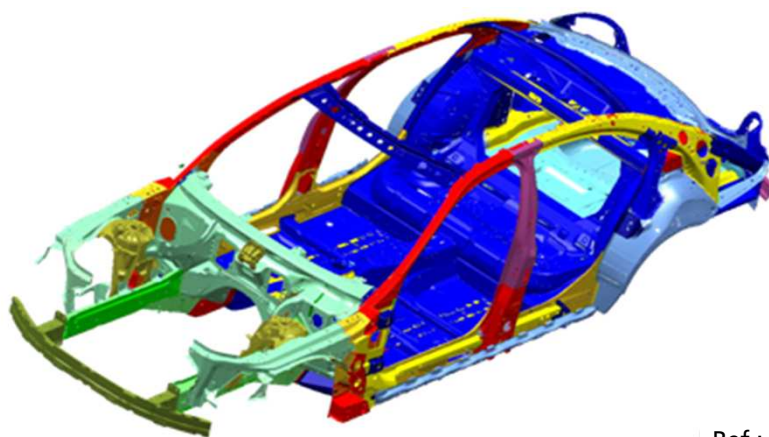
Car Structure in White



DIA 6857e

 **Fraunhofer**
LBF

Karosserierohstruktur

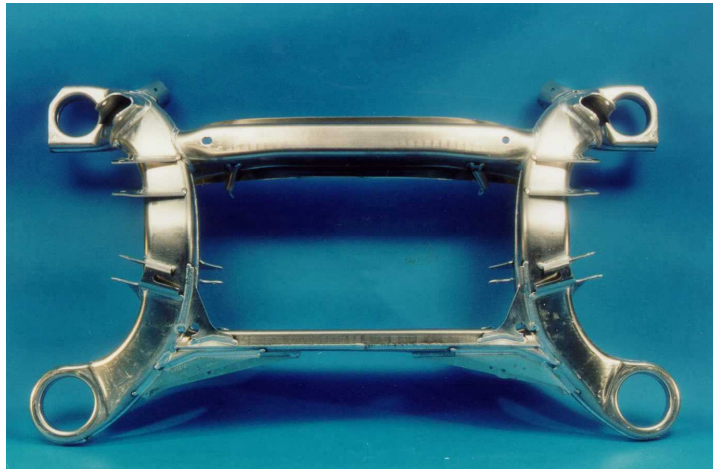


Ref.: BMW AG

DIA 8379e

 **Fraunhofer**
LBF

Welded Aluminium Rearaxle Carrier



DIA 6035e

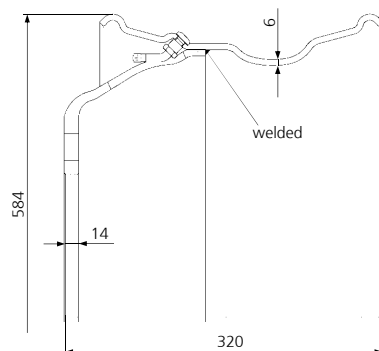
Material: AA 5454 (AlMg3Mn)

Ref.: Daimler AG

Fraunhofer
LBF

Comparison of Welded Steel and Cast Aluminum Truck Wheels (22.5x9.00)

a. Welded steel wheel

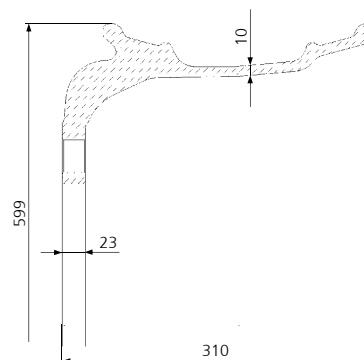


Material: Disk: St 52, Rim: St 37

Weight: 41 kg

Ref.: Hayes Lemmerz Holding GmbH

b. Cast aluminum wheel



Material: G-ALSi7Mg T6 (A 356 T6)

Weight: 27.1 kg

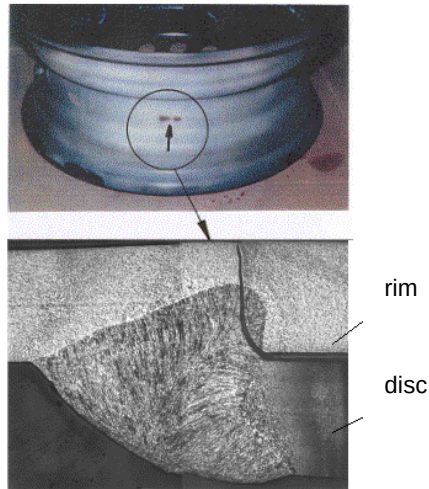
Ref.: Borbet GmbH

DIA 7451e

Fraunhofer
LBF

Fatigue Crack on Welding between Disc and Rim

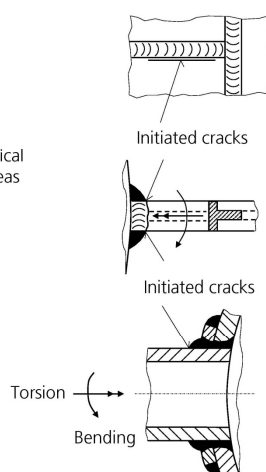
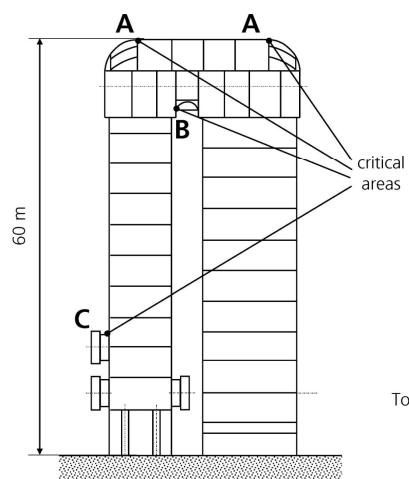
DIA 7302e



Fraunhofer
LBF

Hot-Blast Furnace and Critical Areas

DIA 5521 d = 6240 e



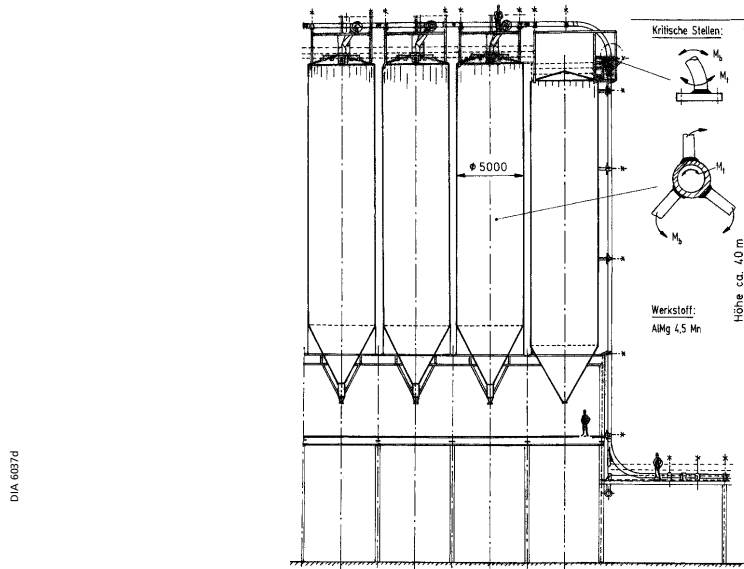
A
Constant amplitude biaxial
loading with constant
principal stress directions

B
Constant amplitude bending
and torsion with changing
principal stress directions

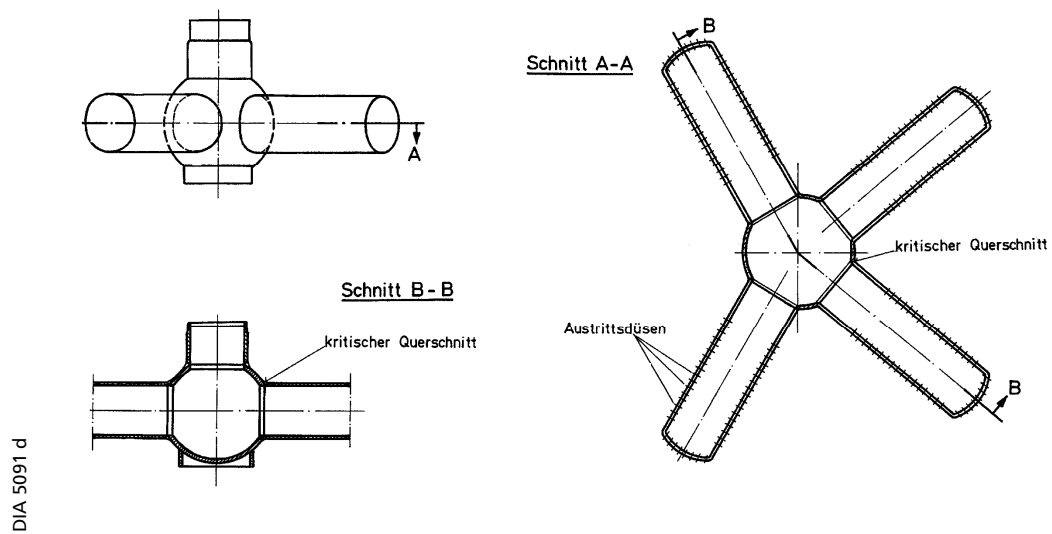
C
Constant amplitude bending
and torsion with changing
principal stress directions

Fraunhofer
LBF

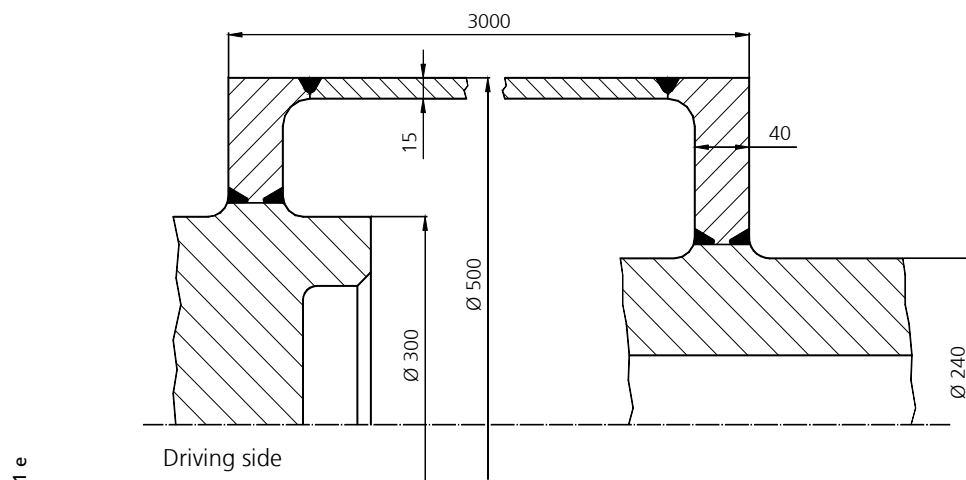
Silos aus Aluminium zur Aufbewahrung von Kunststoffgranulaten



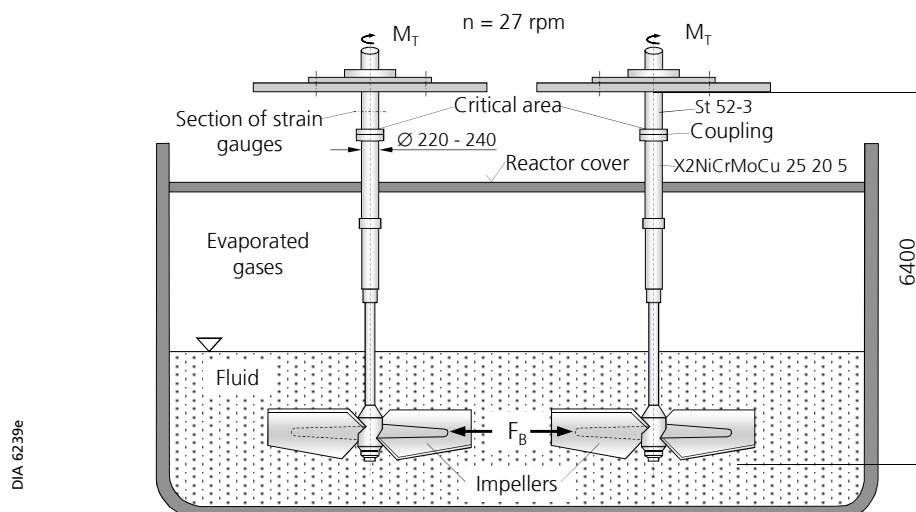
Kritische Bereiche eines Druckentlastungssystems eines Reaktors



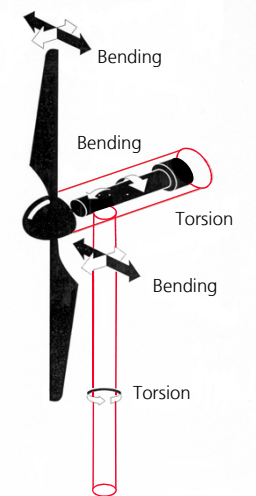
Fatigue Critical Areas of a Welded Stirrer



Stirrers of a Fertilizer Plant

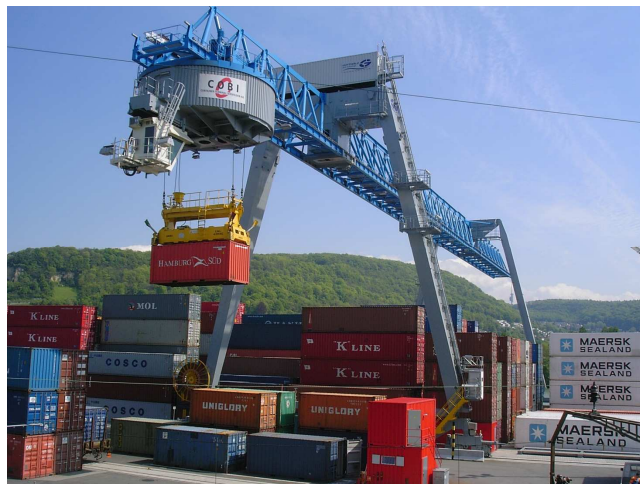


Most Important Loading Components of a Wind Power Plant



DIA 7138e.ppt

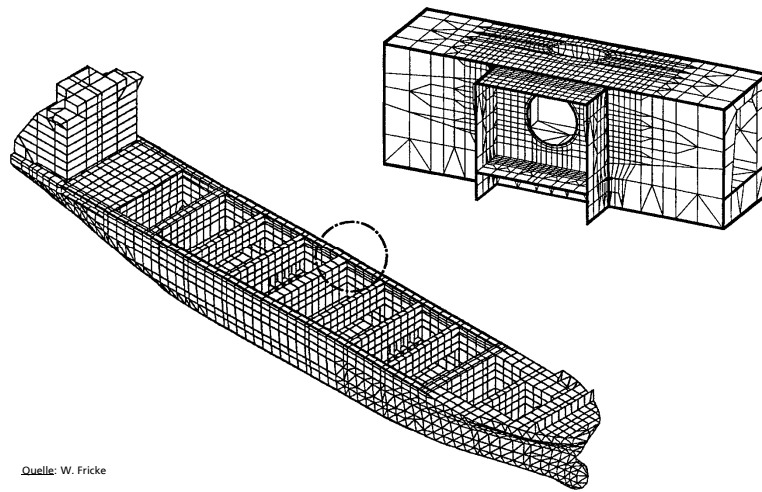
Geschweißte Krankonstruktion



DIA 7652d

Ref.: Gottwald Port Technology GmbH

Schiffstruktur mit Schweißverbindungen und Durchbrüchen in hoch beanspruchten Bereichen

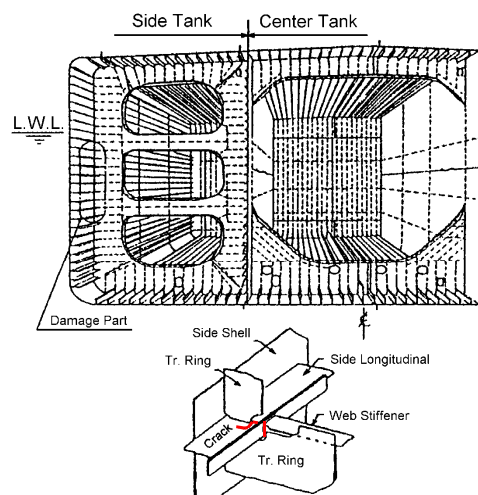


DIA 7640d

Quelle: W. Fricke

Fraunhofer
LBF

Fatigue Damages in Tankers



Ref.: W. Fricke (TUHH)

DIA 7205e

Fraunhofer
LBF

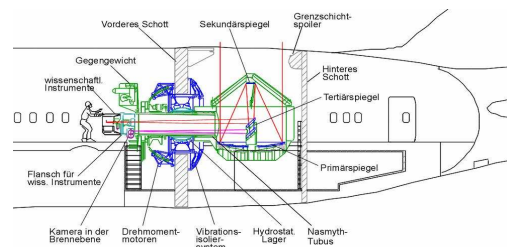
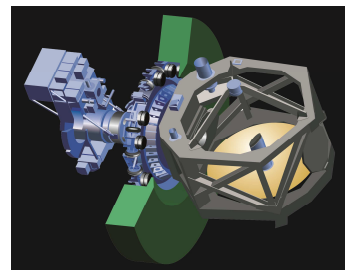
Seto Bridge, Highway and Railway Combinations of Suspension and Cable Stayed Bridges



Ref.: Ch. Miki

Fraunhofer
LBF

Stratospheric Observatory for Infrared Astronomy (SOFIA)

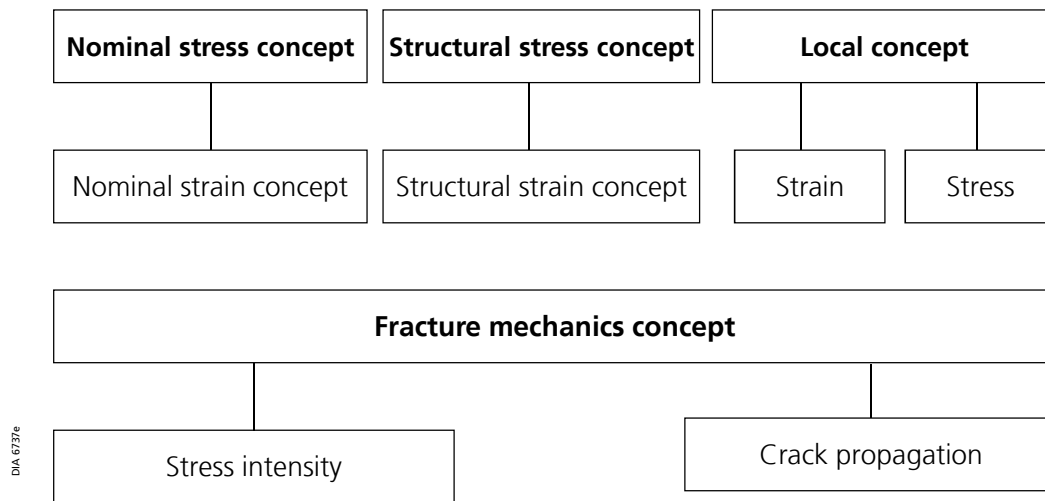


Ref.: NASA, DLR, MAN Technologies

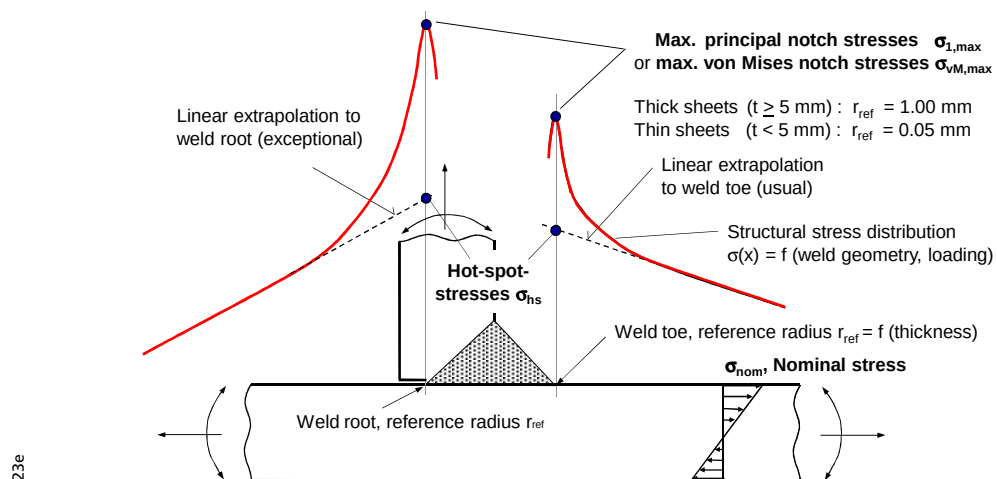
Fraunhofer
LBF

DIA 8680 e

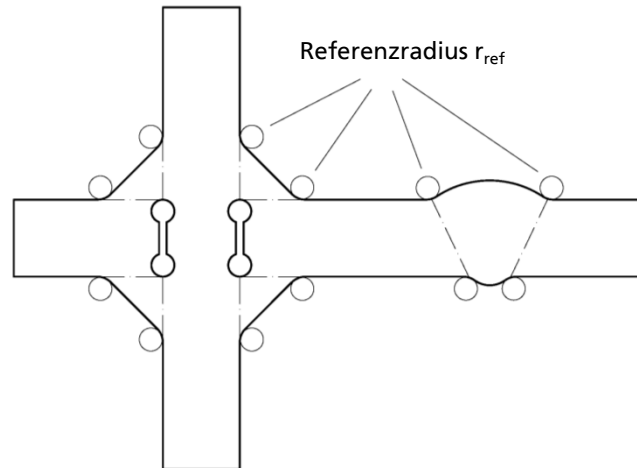
Design Concepts of Structural Durability



Definition of Stresses According to Different Assessment Concepts

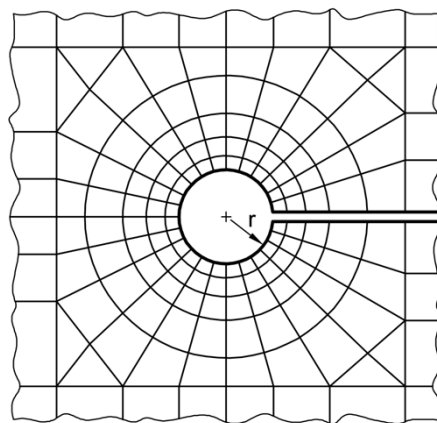


Berechnung von Kerbspannungen mittels des Referenzradius



DIA 7472d

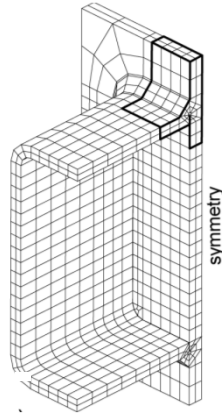
Typisches FE-Netz für eine Schlüssellochkerbe aus Elementen mit quadratischem Ansatz



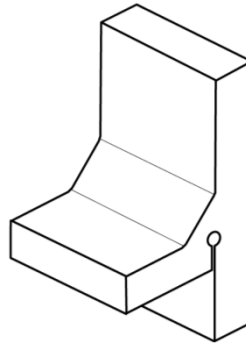
DIA8137d

FE-Modelling of a Fillet-Welded End Joint of a Rectangular Hollow Section

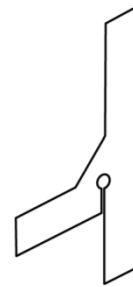
a. Overall model



b. 3D-submodel



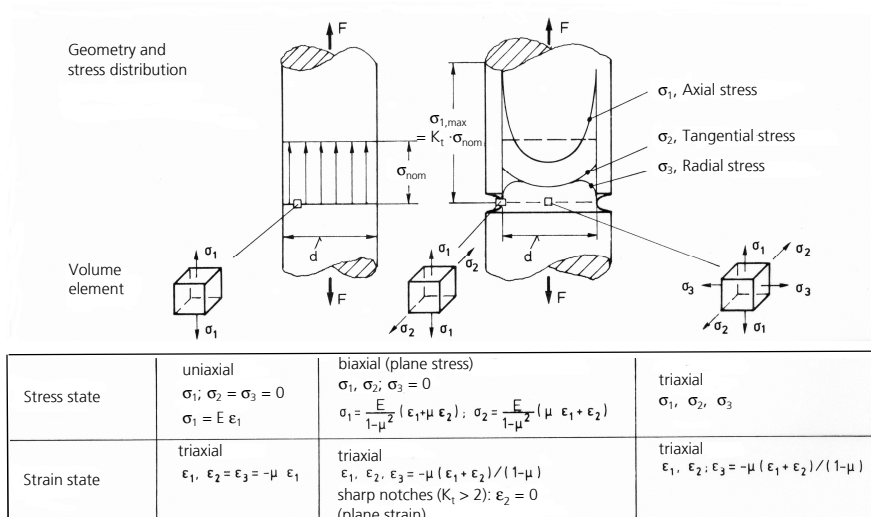
c. 2D-submodel



Ref.: W. Fricke

DIA 8203e

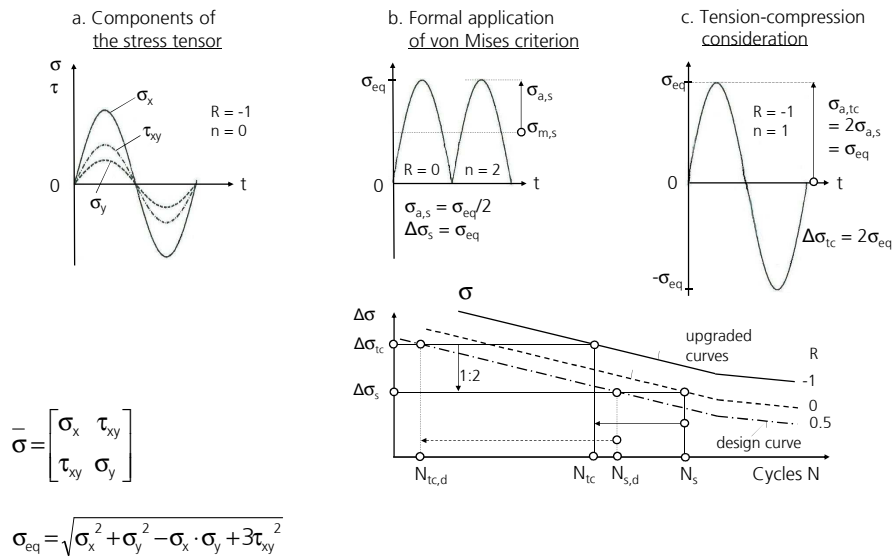
Uni- and Multiaxial Stress-Strain-States



DIA 7137e.ppt

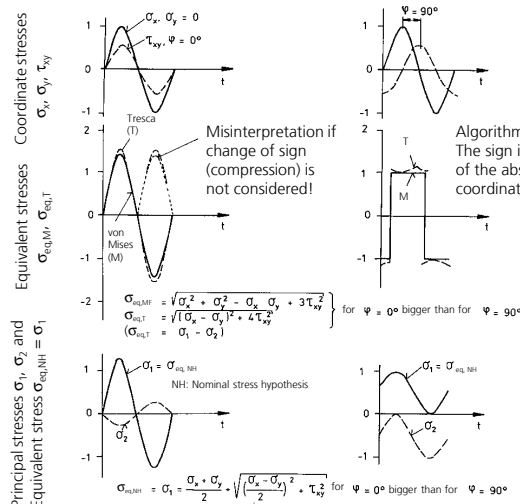
Errors by Formal (Scalar) Application of von Mises Hypothesis

DIA 8099e.ppt



Conventional Strength Hypotheses

DIA 5198e.ppt



Comments for fatigue life assessment:

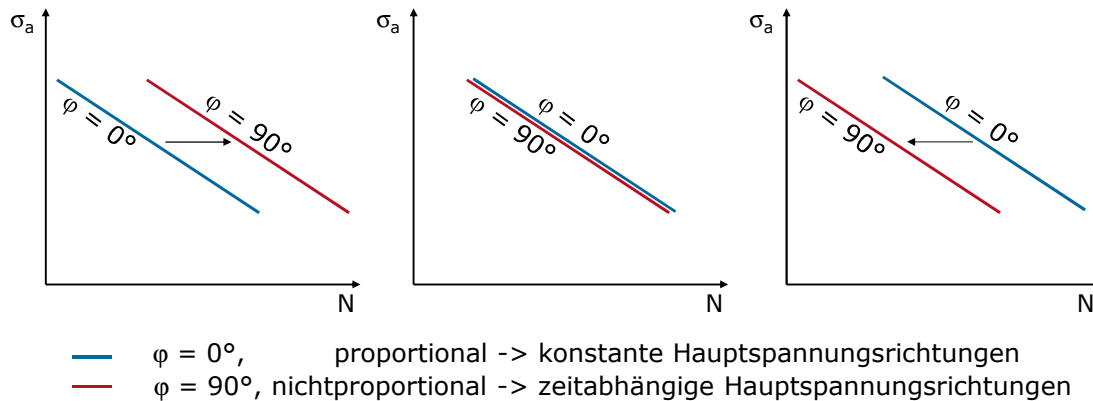
A higher fatigue life by out-of-phase loading is for ductile materials misrepresented. Therefore, need of an appropriate hypothesis.

Einfluss von zeitabhängigen Hauptspannungsrichtungen auf die Lebensdauer

Sprödes (wenig duktiles)
Materialverhalten

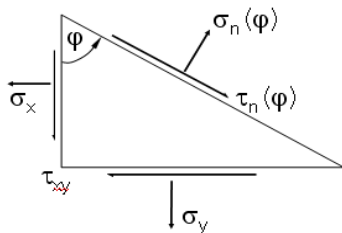
Semi-duktilen
Materialverhalten

Duktilen
Materialverhalten



Spannungen in der Schnittebene und Schadensmechanismen

a. Schnittebenenspannungen



$$\sigma_n(\varphi) = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cdot \cos 2\varphi + \tau_{xy} \cdot \sin 2\varphi$$

$$\tau_n(\varphi) = \frac{\sigma_x - \sigma_y}{2} \cdot \sin 2\varphi - \tau_{xy} \cdot \cos 2\varphi$$

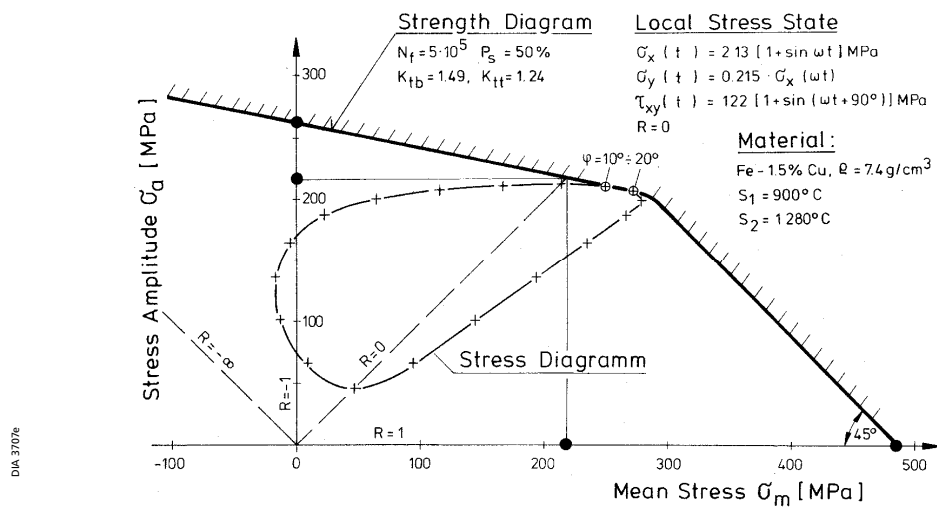
b. Zähigkeitsabhängiger Schadensmechanismus

- $\sigma_n(\varphi)$ für **spröde** Werkstoffe (Aluminiumguss, Gusseisen, Sinterstähle)
- Kombination aus $\sigma_n(\varphi)$ und $\tau_n(\varphi)$ für **semi-duktilen** Werkstoffe (Aluminiumknetlegierung, Stahlguss)
- $\tau_n(\varphi)$ für **duktilen** Werkstoffe (z.B. Baustähle)

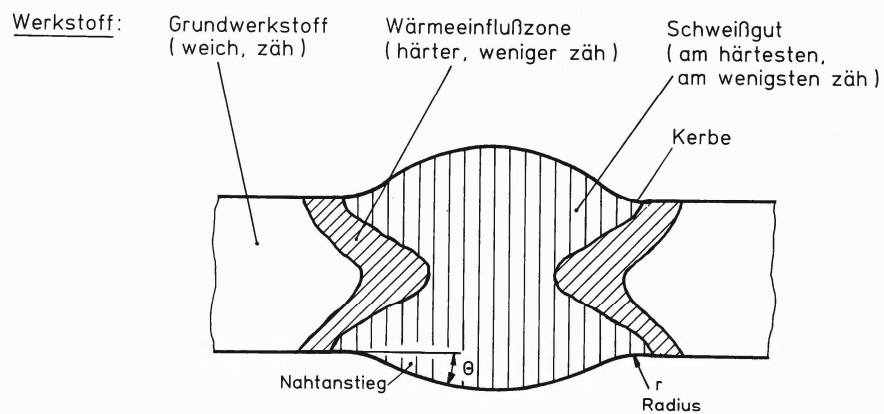
c. Rechenalgorithmus

- Maximale Normalspannung in der kritischen Schnittebene
- Spannung als Kombination aus Normal- und Schubspannung in der kritischen Schnittebene
- Integraler Wert aus der Schubspannung über alle Schnittebenen

Correlation of the Calculated Stresses and Fatigue Strength



Verbundwerkstoff Schweißnaht



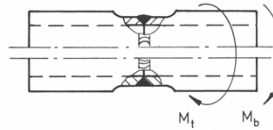
Geometrie: Nahtübergangskerbe, $K_t = f(r, \theta, \text{Belastungsart})$

DIA 5185d.ppt



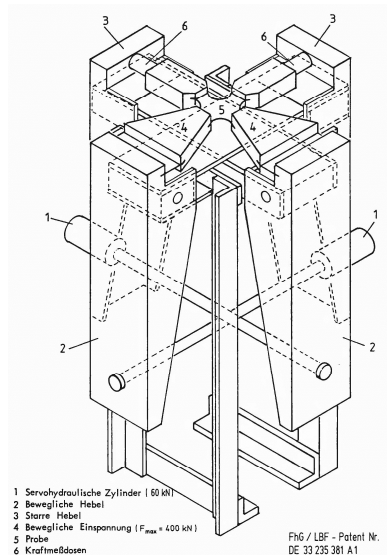
- b. Rohr - Flansch - Verbindung
-
- $b_1:$
- $b_2:$

- c. Rohr - Rohr - Verbindung

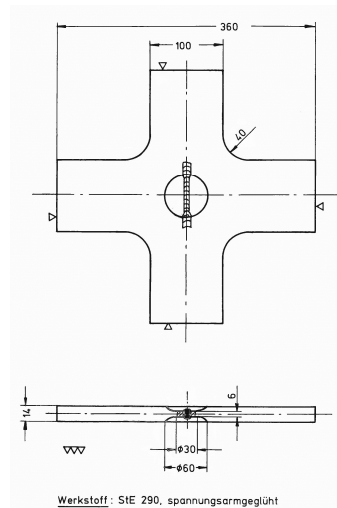


- 

DIA 8702 d

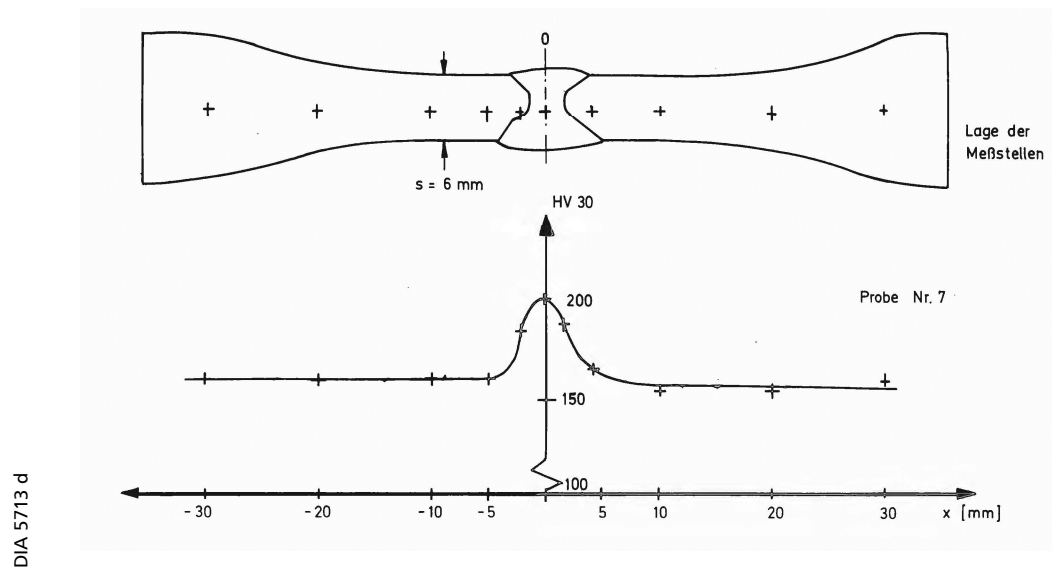


FhG / LBF - Patent Nr.
DE 33 235 381 A1

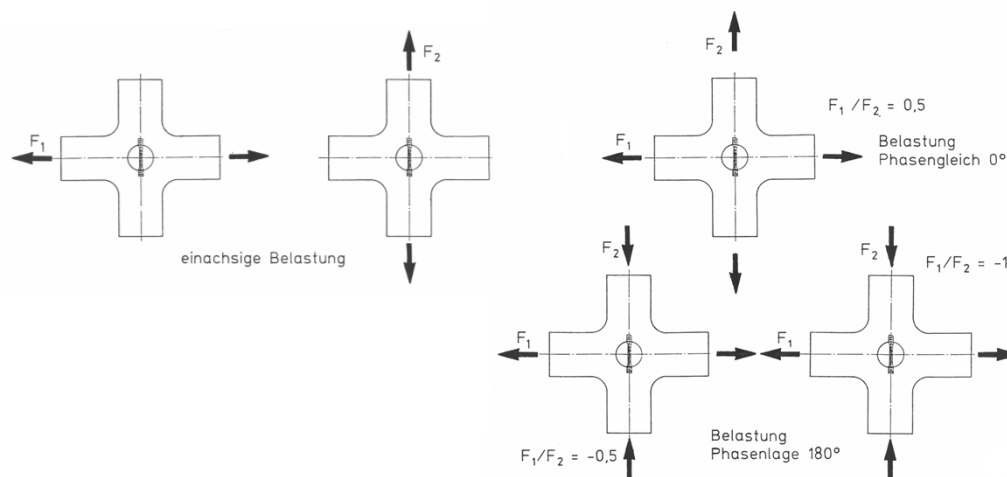


Werkstoff: StE 290, spannungsarmgeglüht

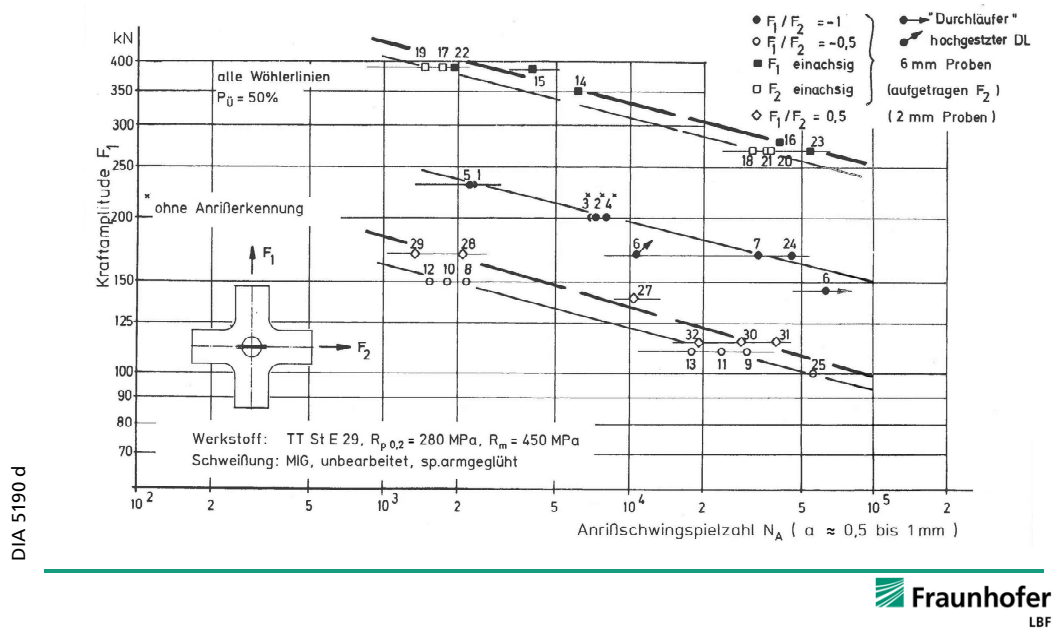
Härteverteilung quer zur Schweißnaht der Kreuzproben



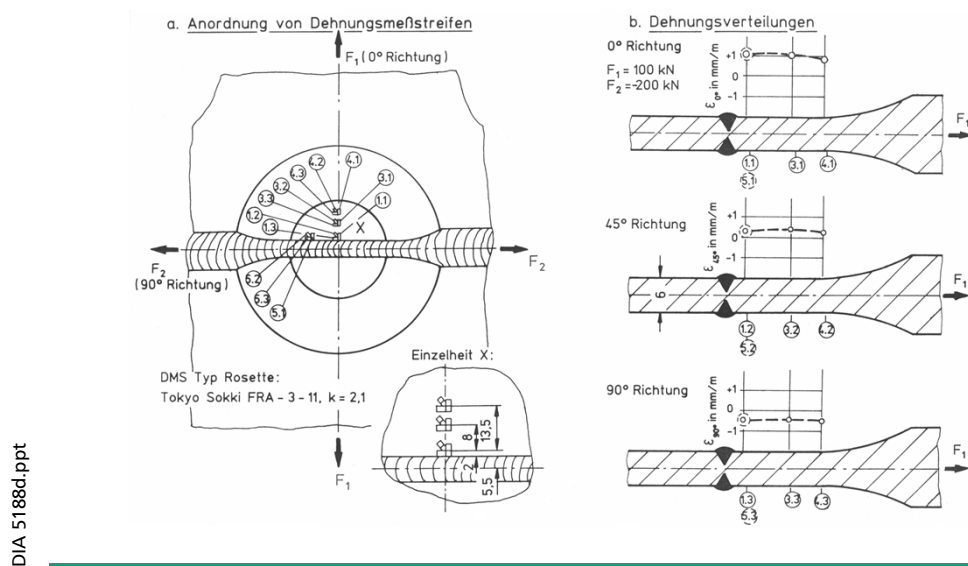
Untersuchte Belastungsverhältnisse



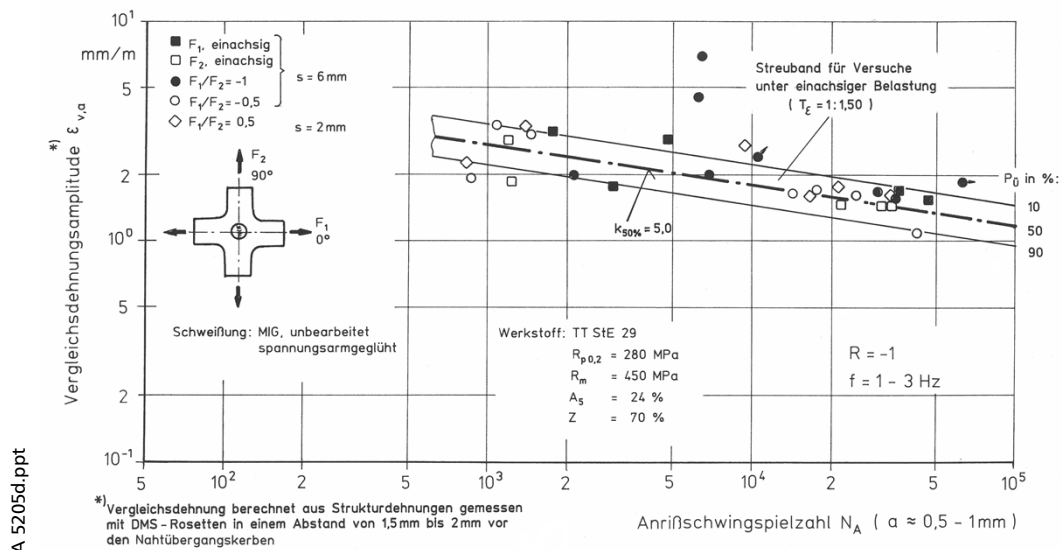
Uni- und biaxiale Schwingbelastung der geschweissten Kreuzproben



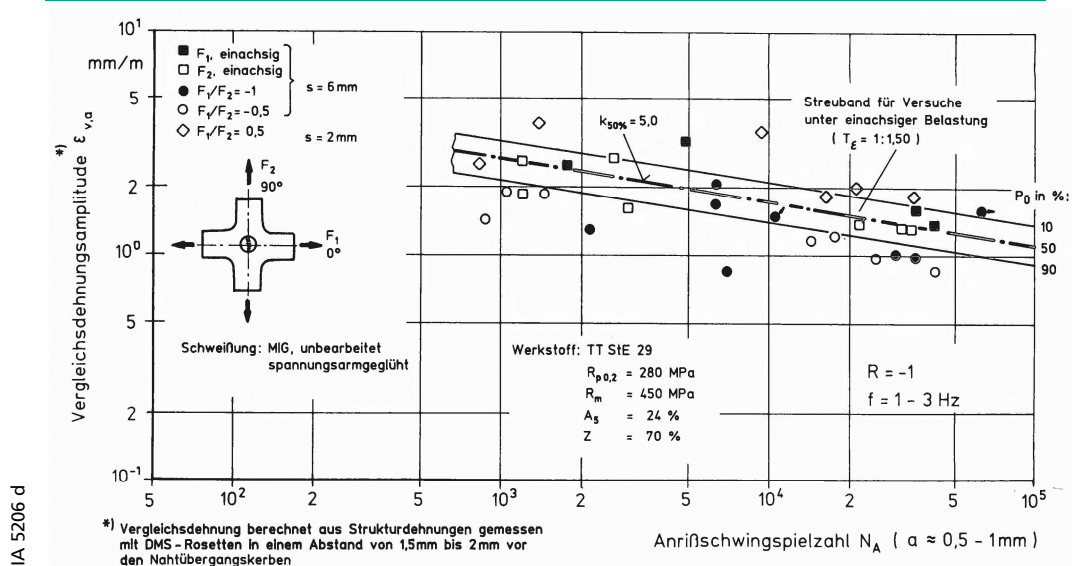
Anordnung von Dehnmessstreifen und Beispiel einer Dehnungsmessung



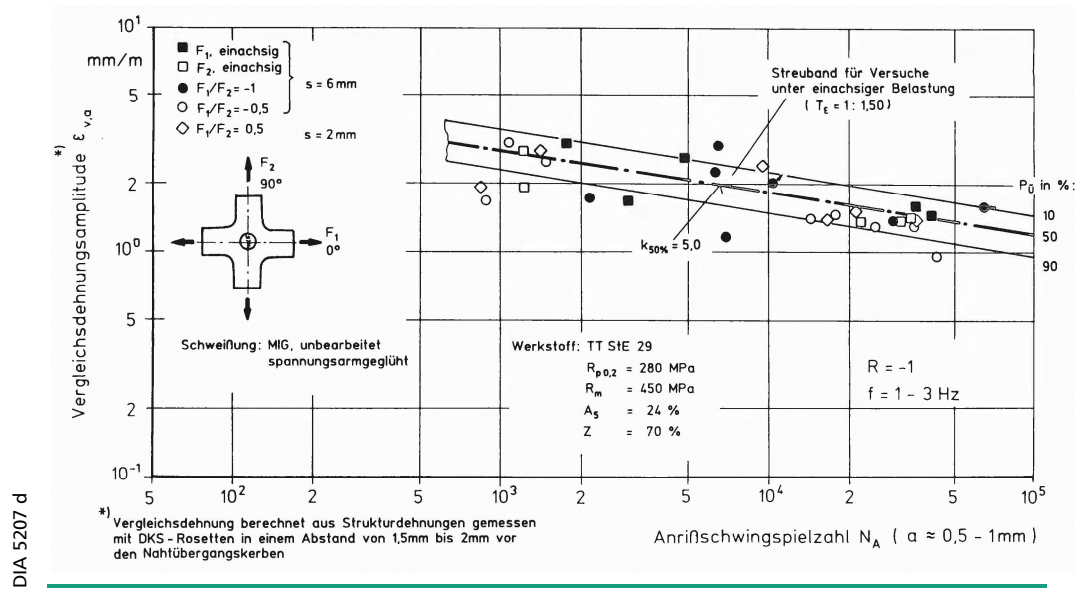
Bewertung der Ergebnisse nach der Gestaltänderungsenergiehypothese



Bewertung der Ergebnisse nach der Normalspannungshypothese

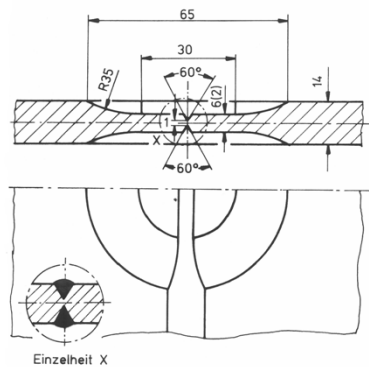


Bewertung der Ergebnisse nach der Normaldehnungshypothese



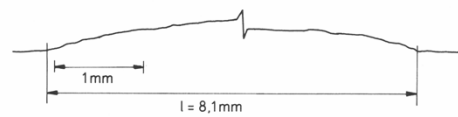
Angaben zur Probengeometrie

a. Probengeometrie

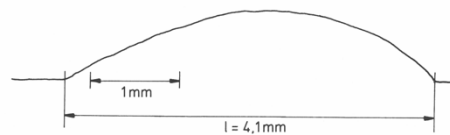


b. Schweißnahtprofile

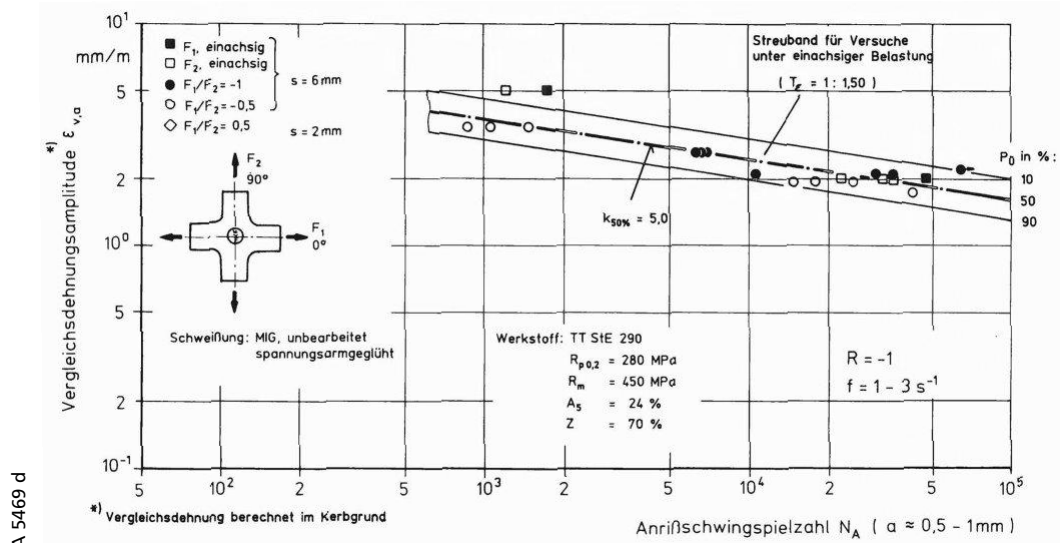
Kontur einer Probe mit $s = 6 \text{ mm}$



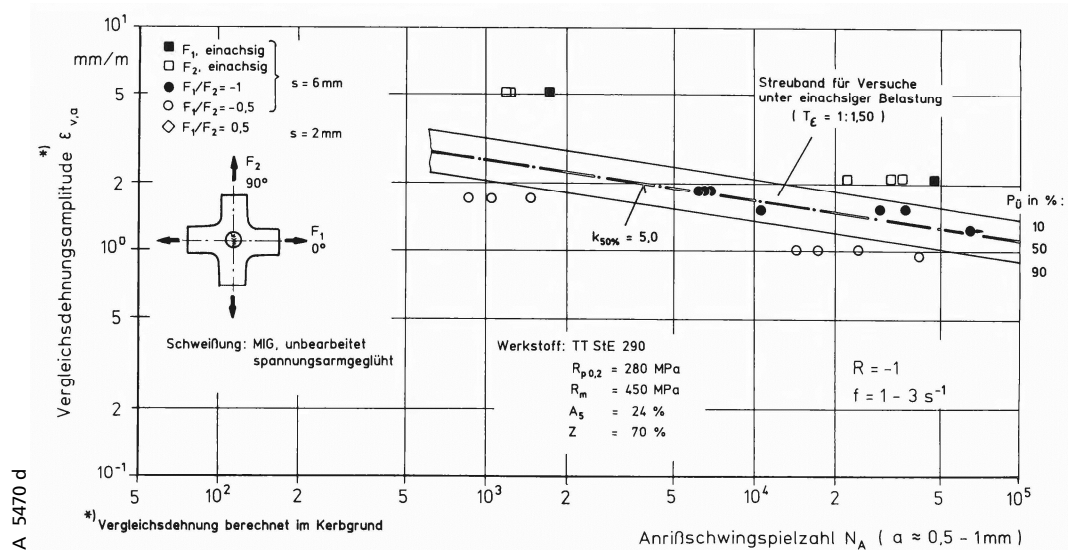
Kontur einer Probe mit $s = 2 \text{ mm}$



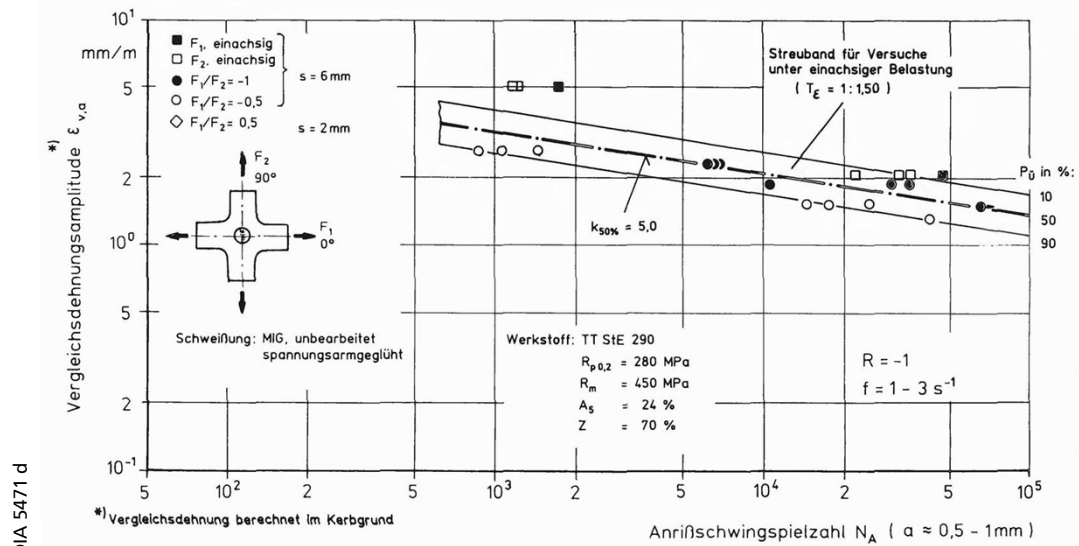
Bewertung der Ergebnisse nach der Gestaltsänderungsenergiehypothese



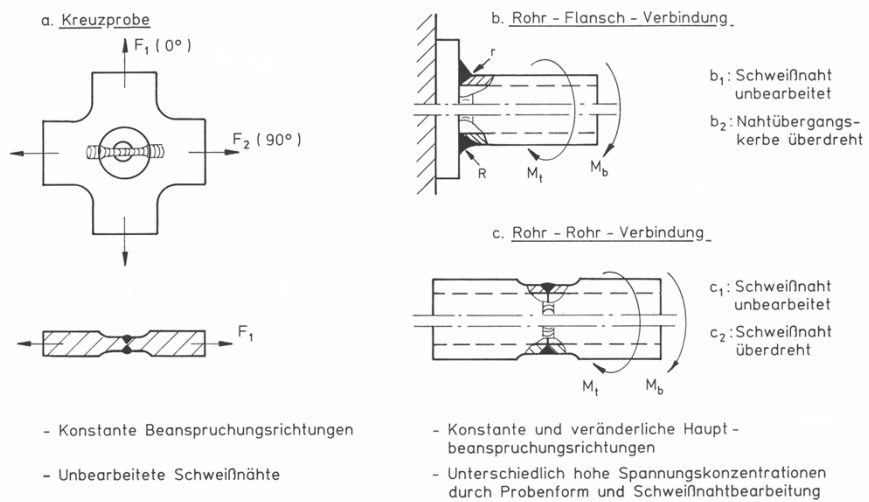
Bewertung der Ergebnisse nach der Normalspannungshypothese



Bewertung der Ergebnisse nach der Normaldehnungshypothese

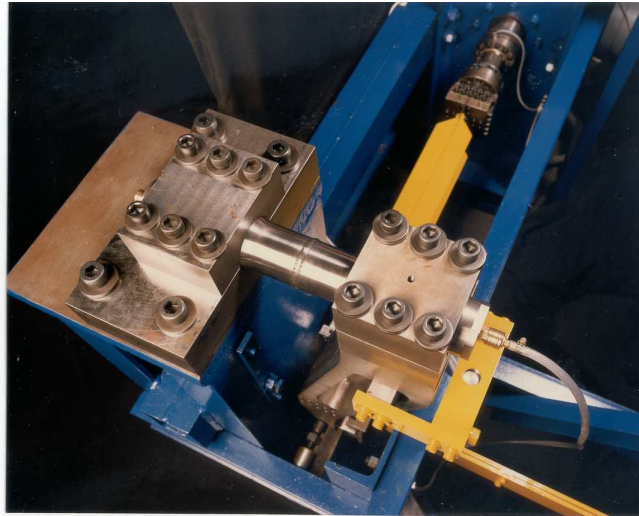


Probenformen zur Untersuchung des Einflusses von mehrachsigen Beanspruchungen auf das Festigkeitsverhalten von Schweißverbindungen



Mehrachsigkeit - Winkelprüfstand

DIA 5139d



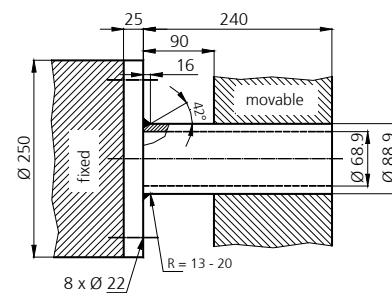
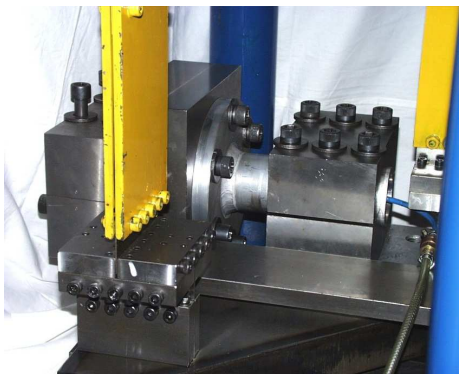
Fraunhofer
LBF

Test Rig and Specimen Geometry for Aluminium Connections

a. Experimental setup for biaxial testing

b. Flange-tube specimen and clamping

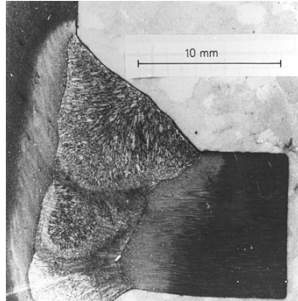
DIA 6713e.ppt



Fraunhofer
LBF

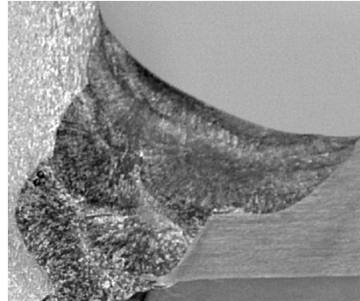
Geometry of the Welds

a. Steel St E 460



$r_m = 0,45 \text{ mm}$, $\theta = 45^\circ$
 $K_{tb} = 3,93$, $K_{tt} = 1,85$

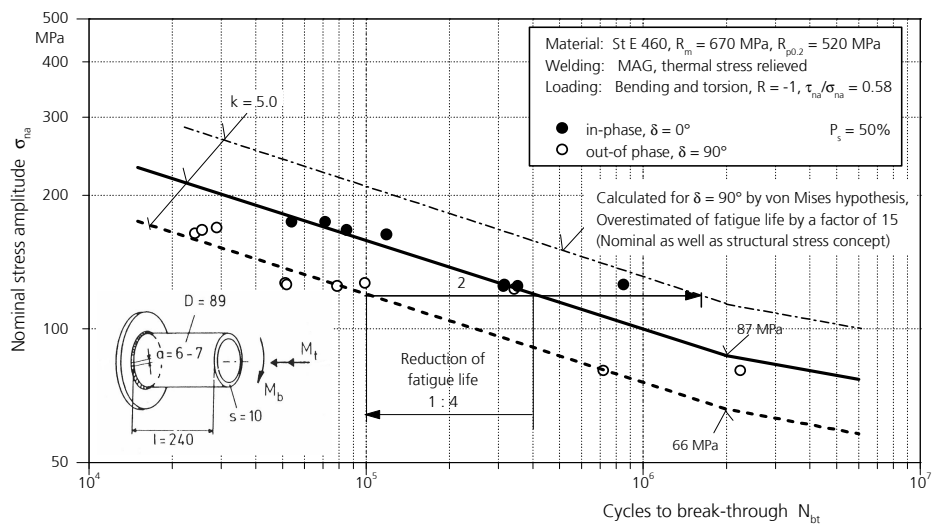
b. Aluminium AlSi1MgMn T6



$r_m = 17,0 \text{ mm}$, $\theta = 38^\circ$
 $K_{tb} = 1,62$, $K_{tt} = 1,14$

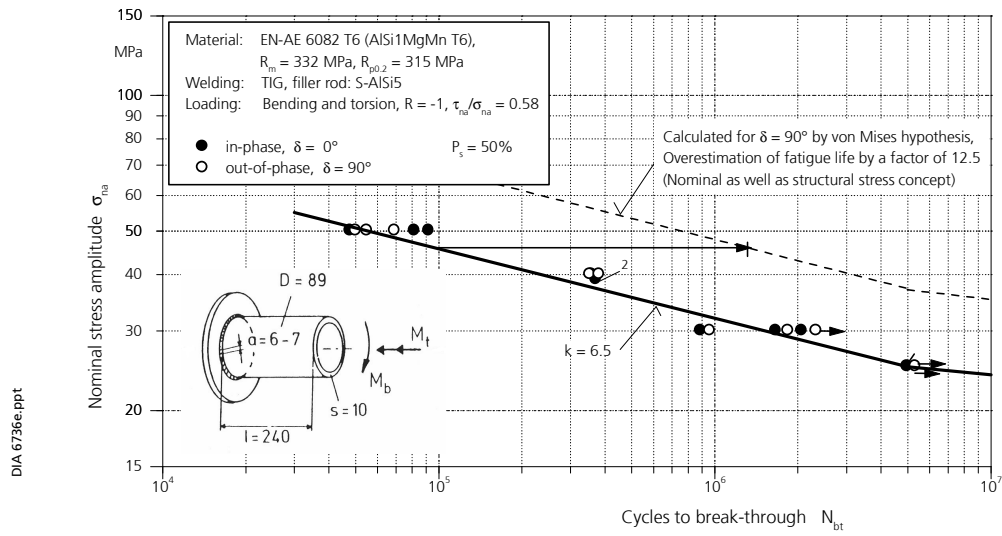
DIA 6763e

Multiaxial Fatigue Behaviour of Welded Steel Joints under Constant Amplitude Loading



DIA 6659e.ppt

Multiaxial Fatigue Behaviour of Welded Aluminium Joints under Constant Amplitude Loading



Effective Equivalent Stress Hypothesis (EESH)

$$F(\delta) = \frac{1}{\pi} \int_0^\pi f_p(\varphi) d\varphi \quad (1)$$

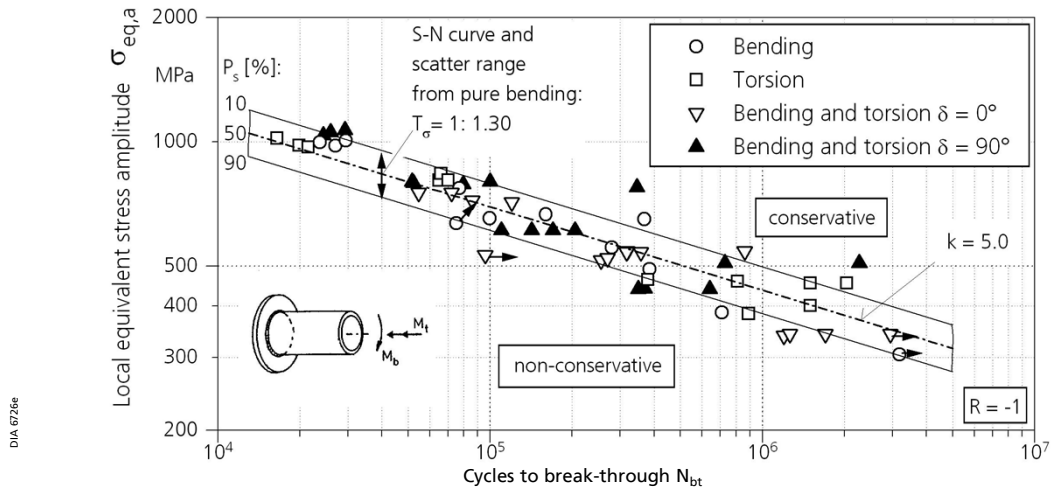
$$f_p(\varphi) = \tau_n(\varphi) \quad (2)$$

$$f_p(\varphi) = \frac{1}{L_s} \sum_{i=1}^{L_s} \tau_{n,i}(\varphi) \quad (3)$$

$$\sigma_v(\delta) = \sigma_v(\delta = 0^\circ) \cdot \frac{F(\delta)}{F(\delta = 0^\circ)} \quad (4)$$

$$\sigma_v(\delta = 0^\circ) = \sqrt{\sigma_x^2 + \sigma_y^2 - \sigma_x \cdot \sigma_y + 3 \tau_{xy}^2} \quad (5)$$

Application of the Effective Equivalent Stress Hypothesis for As-Welded Flange-Tube Connections under Combined Constant Amplitude Loading



Hypothesis of the Combined Normal and Shear Stress (KoNoS)

$$S(\varphi, \delta) = \sqrt{\sigma_n^2(\varphi, \delta) + 3\tau_n^2(\varphi, \delta)} \quad (1)$$

$$\sigma_v(\delta) = \sigma_v(\delta = 0^\circ) \cdot \frac{S(\varphi^{**}, \delta)}{S(\varphi^*, \delta = 0^\circ)} \quad (2)$$

$$\sigma_v(\delta = 0^\circ) = \sqrt{\sigma_x^2 + \sigma_y^2 - \sigma_x \cdot \sigma_y + 3\tau_{xy}^2} \quad (3)$$

Application of the Hypothesis KoNoS for Tube-to-Plate Weldings under Constant Amplitude Loading

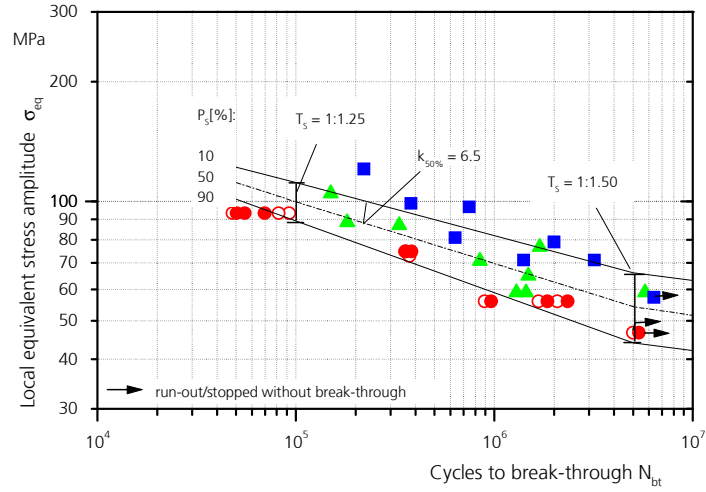
Material: AlSi1MgMn T6
EN-AW 6082 T6
 $R_{p0.2} = 315 \text{ MPa}$
 $R_m = 332 \text{ MPa}$

Welding: TIG
Filler rod: S-AlSi5

Test series: Flange-tube
 $R = -1$
Constant amplitude loading

▲ Bending
■ Torsion
○ Bending and Torsion
($\delta = 0^\circ, T_d/S_d = 0.58$)
● Bending and Torsion
($\delta = 90^\circ, T_d/S_d = 0.58$)

DIA 7523e



Fraunhofer
LBF

Introduction of Local Normal and Shear Stresses into the Gough-Pollard Equation According to Different Hypotheses

Modified Gough-Pollard Equation: $\left(\frac{\Delta\sigma_{Load}}{\Delta\sigma_{SN}(N)} \right)^{k_\sigma} + \left(\frac{\Delta\tau_{Load}}{\Delta\tau_{SN}(N)} \right)^{k_\tau} \leq D_{MA} \quad k_\sigma = k_\tau = 2$

Stress tensor: $\Delta\bar{\sigma}_{Load} = \begin{bmatrix} \Delta\sigma_{x,Load} & \Delta\tau_{xy,Load} \\ \Delta\tau_{xy,Load} & \Delta\sigma_{y,Load} \end{bmatrix}$

Shear stresses: $\Delta\tau_{load} = \Delta\tau_{xy,load}$
with $\Delta\tau_{SN}(N)$ from pure torsion

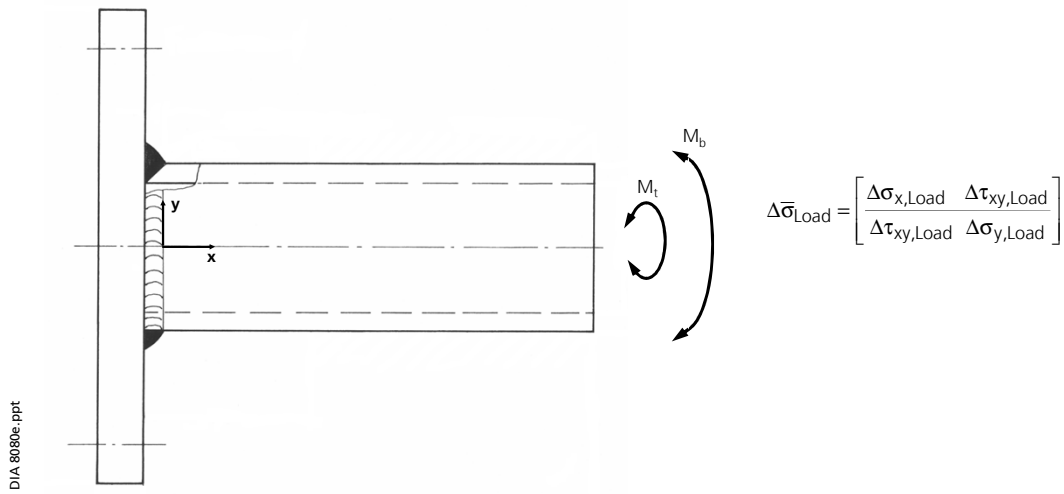
Principal stress: $\Delta\sigma_{Load} = \Delta\sigma_{Load,1} = \Delta\sigma_{x,Load}$
with $\Delta\sigma_{SN}(N)$ according to PSH

von Mises: $\Delta\sigma_{Load} = \Delta\sigma_{Load,von \text{ Mises}} = \sqrt{\Delta\sigma_{x,Load}^2 + \Delta\sigma_{y,Load}^2 - \Delta\sigma_{x,Load} \cdot \Delta\sigma_{y,Load}}$
with $\Delta\sigma_{SN}(N)$ according to von Mises

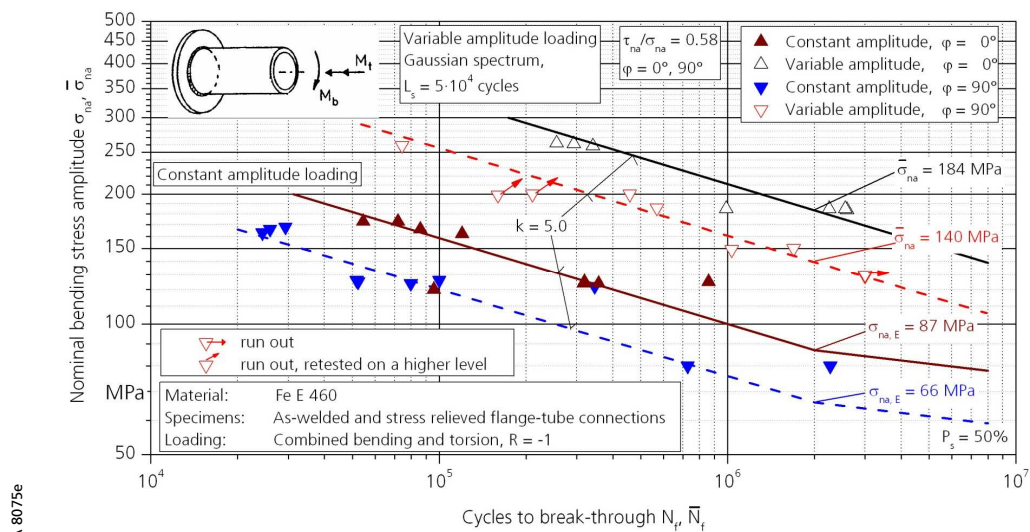
DIA 8070e.ppt

Fraunhofer
LBF

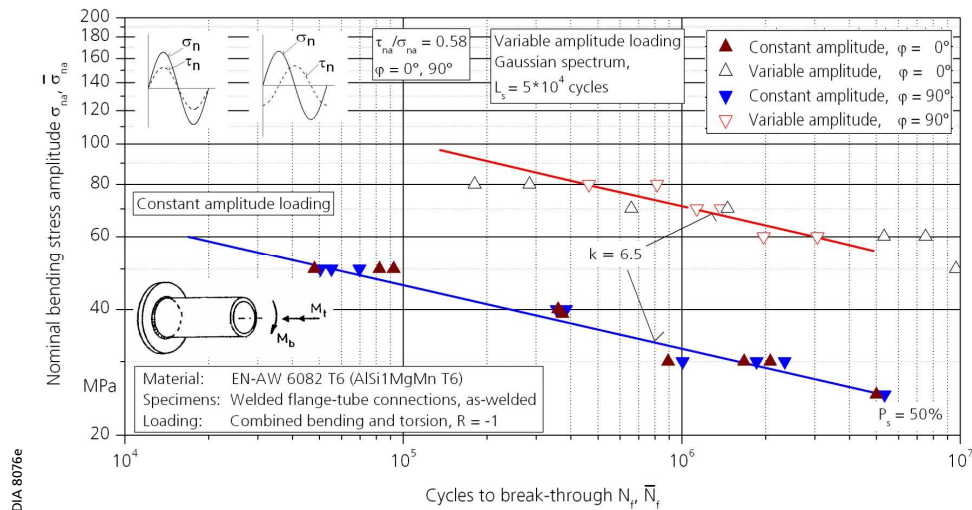
Coordinate System for Normal and Shear Stress Components



Fatigue Testing Results Obtained with As-Welded Steel Specimens under Combined Bending and Torsion ($\varphi = 0^\circ$ and 90°)

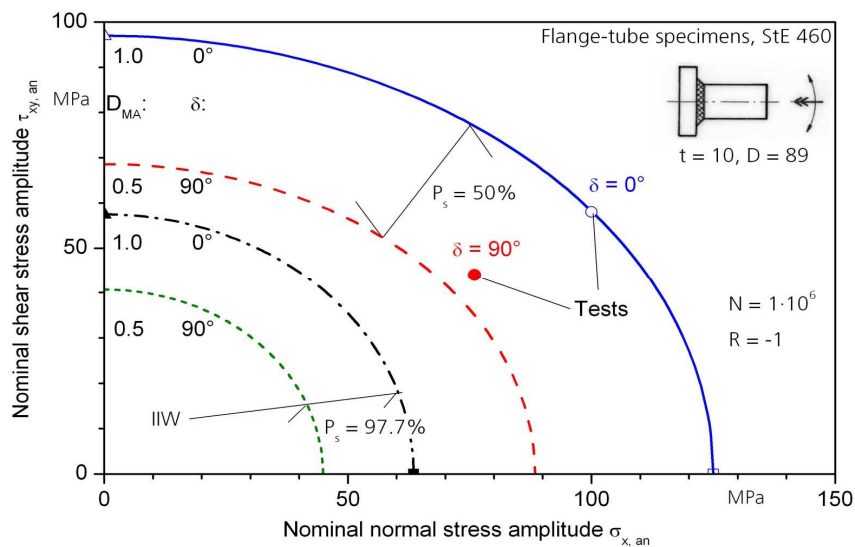


Fatigue Testing Results Obtained with As-Welded Aluminium Specimens under Combined Bending and Torsion ($\varphi = 0^\circ$ and 90°)



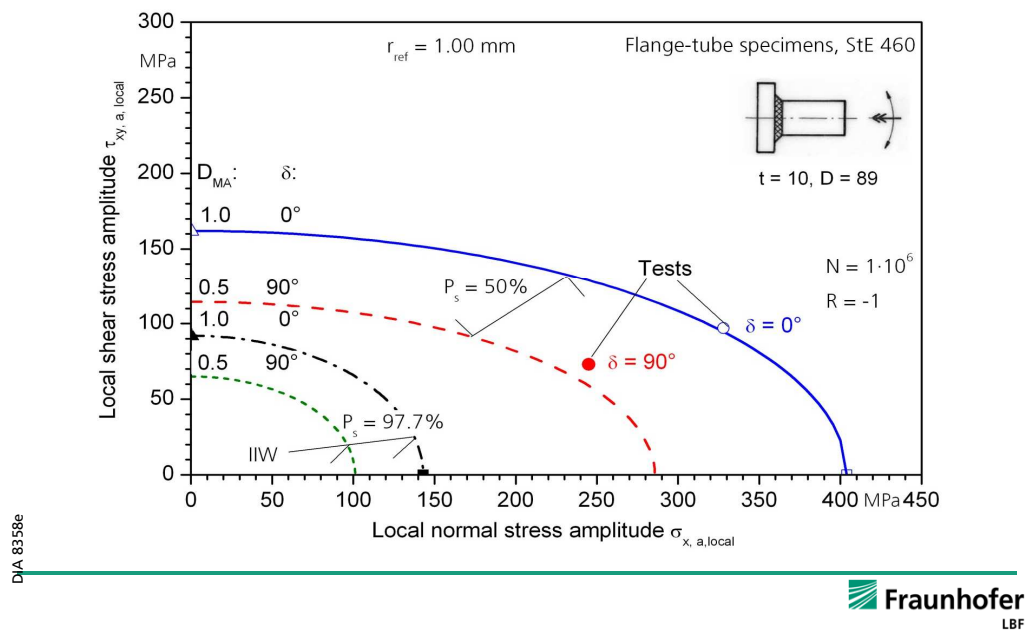
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Assessment of Fusion Welded Thick Steel Flange-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Nominal Stress System)

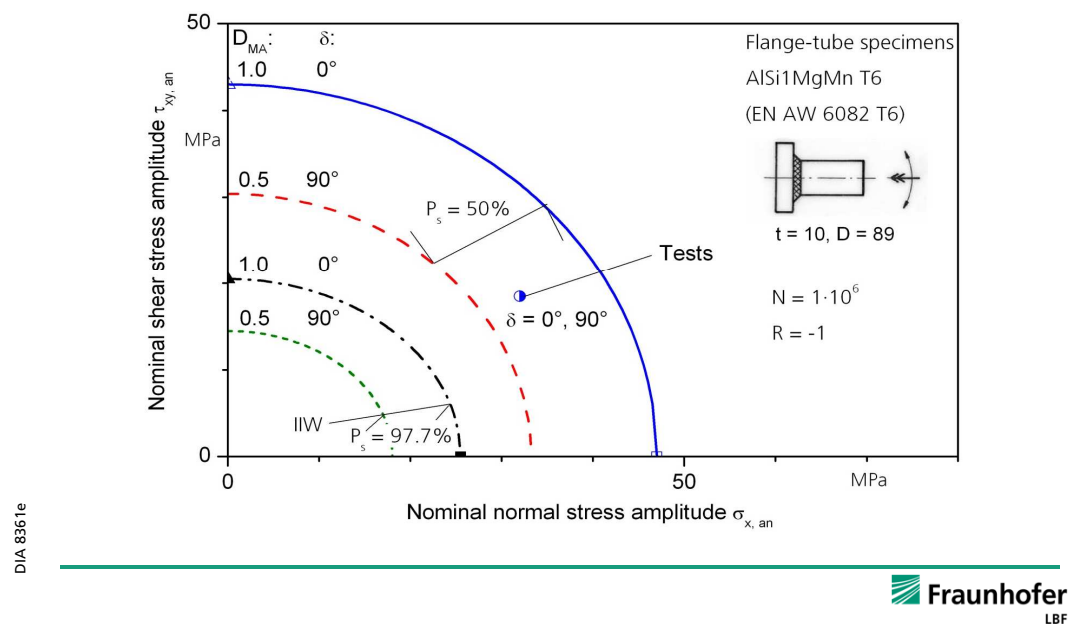


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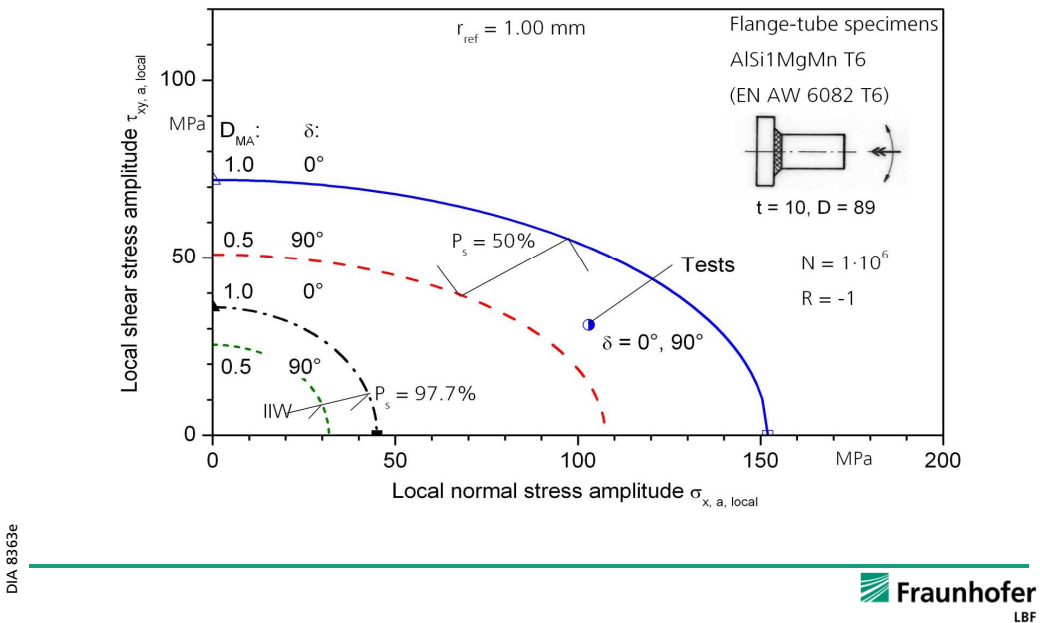
Assessment of Fusion Welded Thick Steel Flange-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



Assessment of Fusion Welded Thick Aluminium Flange-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Nominal Stress System)



Assessment of Fusion Welded Thick Aluminium Flange-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



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Recommended Assessment Procedure for Multiaxial Loading (2010)

$$\left(\frac{\Delta \sigma_{load}}{\Delta \sigma_{SN}(N)} \right)^{k_\sigma} + \left(\frac{\Delta \tau_{load}}{\Delta \tau_{SN}(N)} \right)^{k_\tau} \leq D_{MA}$$

$k_\sigma = k_\tau = 2$	Exponents of the Gough-Pollard equation
$D_{MA} = 1.0$	For constant principal stress directions, in-phase (proportional) loading for steel (Eurocode 3, IIV) and aluminium
$D_{MA} = 0.5$	For changing principal stress directions, out-of-phase (unproportional) loading for steel and ductile aluminium alloys*)
$\Delta \sigma_{load} = 2 \cdot \sigma_{a,load}$ $\Delta \tau_{load} = 2 \cdot \tau_{a,load}$	Nominal, hot-spot or local stress components of the biaxial stress state induced by external loading
$\Delta \sigma_{SN}(N) = 2 \cdot \sigma_{a,SN}(N)$ $\Delta \tau_{SN}(N) = 2 \cdot \tau_{a,SN}(N)$	Endurable nominal, hot-spot or local stresses at N cycles under pure axial loading (or bending) and pure torsion

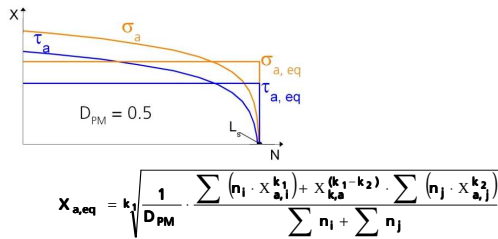
*)For semi-ductile aluminium alloys $D_{MA} = 1.0$

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Prerequisites and Equations for the Assessment of Multiaxial Spectrum Loading

1 Cumulative damage

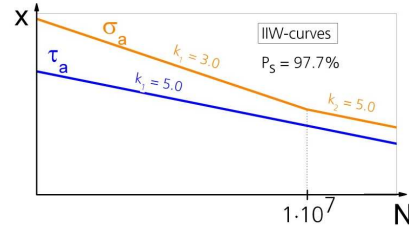
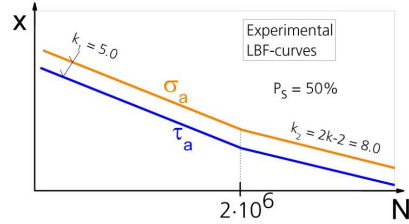


2 Multiaxial assessment

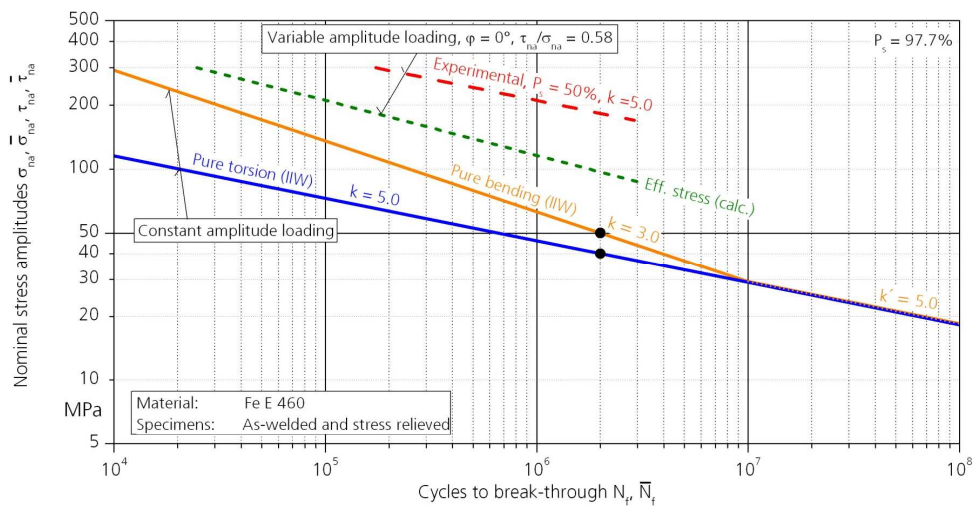
$$\left(\frac{\sigma_{a,eq}}{\sigma_{a,SN}(N)} \right)^2 + \left(\frac{\tau_{a,eq}}{\tau_{a,SN}(N)} \right)^2 \leq D_{MA}$$

Material	Proportional loading D_{MA}	Non-proportional loading D_{MA}
Steel / Aluminium	1.0	0.5

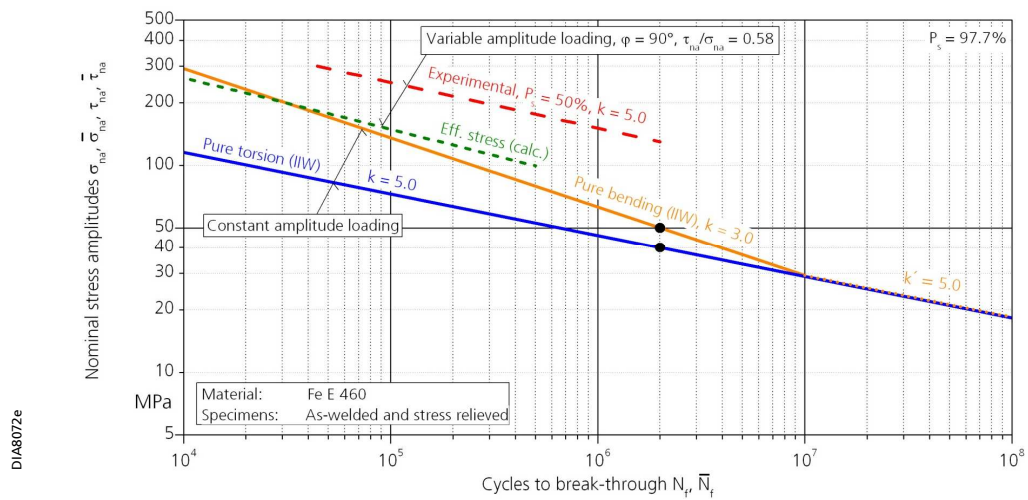
3 SN curves



Evaluation of Multiaxial Spectrum Loading on As-Welded Steel Joints with IIW-SN-Lines

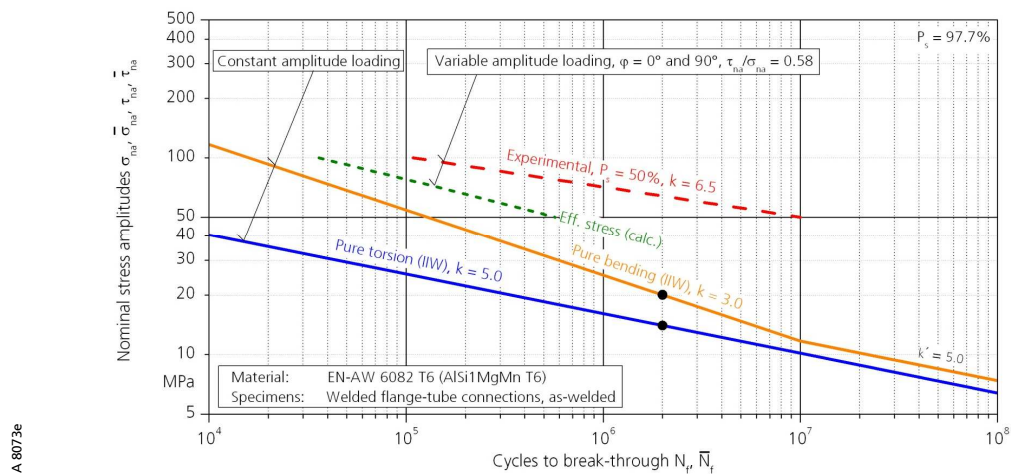


Evaluation of Multiaxial Spectrum Loading on AS-Welded Steel Joints with IIW-SN-Lines



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Evaluation of Multiaxial Spectrum Loading on AS-Welded Steel Joints with IIW-SN-Lines



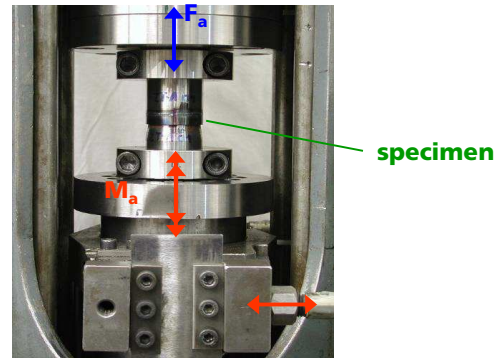
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Biaxial Testing of Overlapped Laserbeam-Welded Tube-Tube Specimens

Test conditions:

- Constant and variable amplitudes under load control
- Load ratio: $R = -1$
- Test frequency: 20 Hz
- Laboratory conditions:
In air at room temperature

Biaxial test rig:



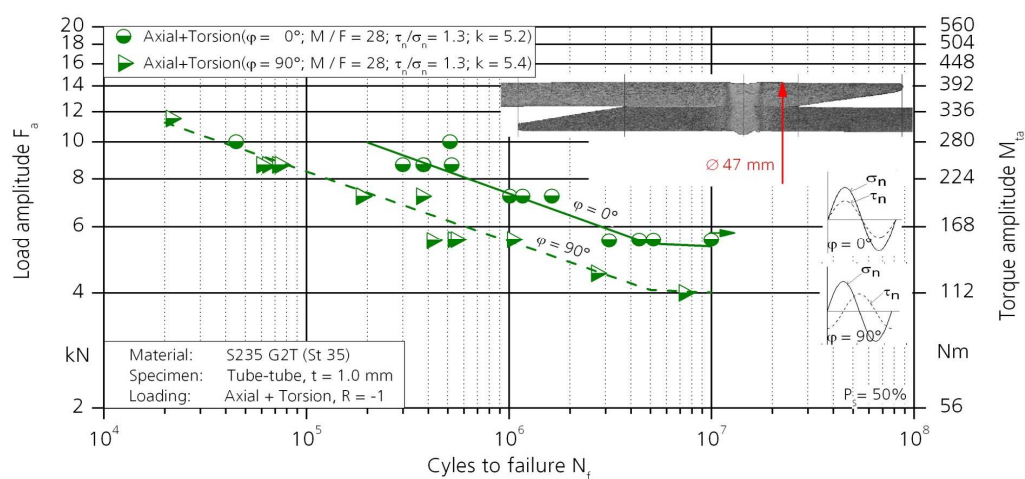
$$F_{a, \max} = 80 \text{ kN}$$

$$M_{a, \max} = 3 \text{ kNm}$$

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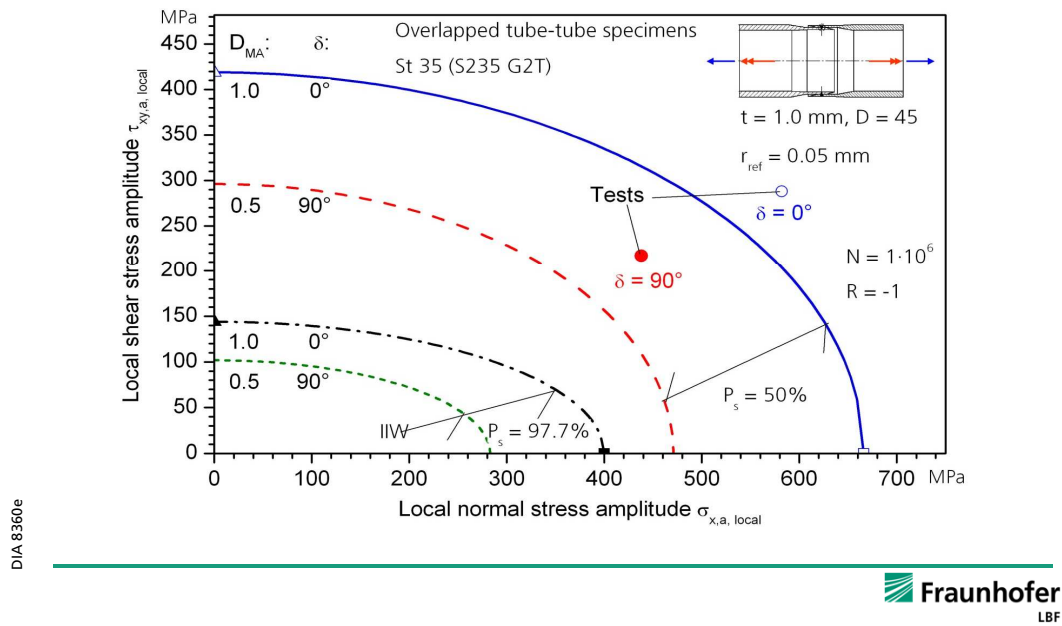
Laserbeam-Welded Overlapped Steel Tube-Tube Specimens under Constant Amplitude Multiaxial Loading



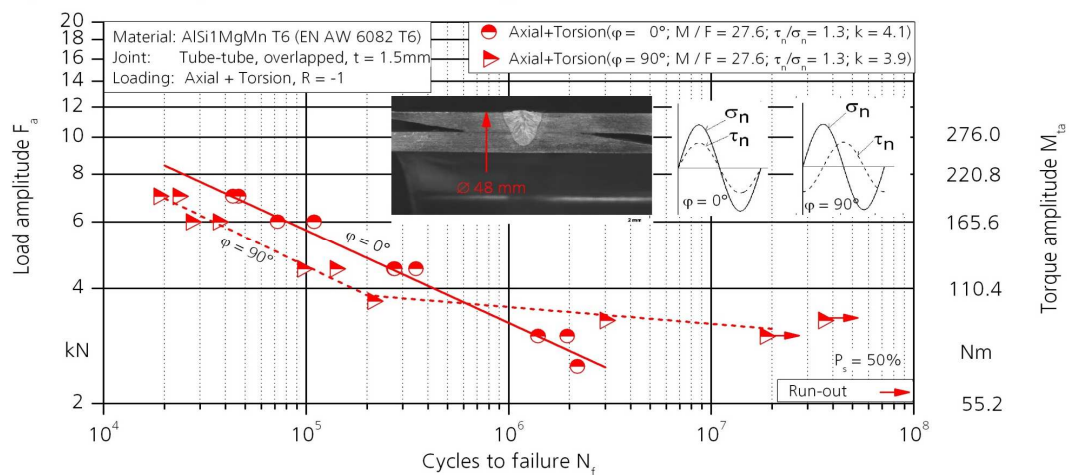
DIA 8302 e

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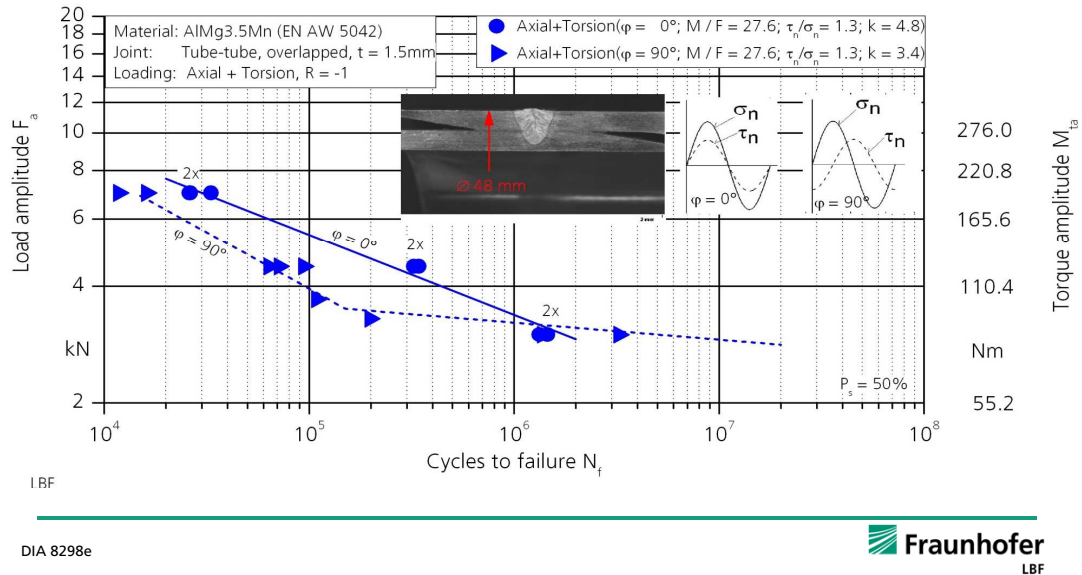
Assessment of Laserbeam-Welded Thin Steel Tube-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



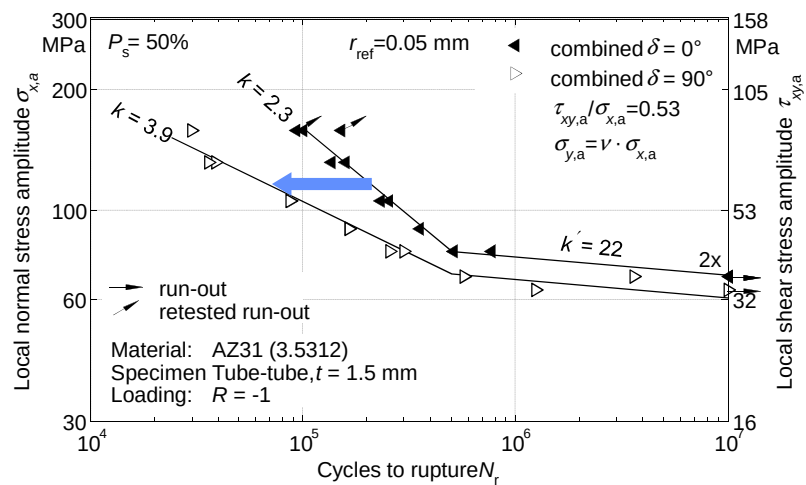
Laserbeam-welded Overlapped Aluminium Tube-Tube Specimanes under Constant Amplitude Multiaxial Loading



Laserbeam-welded Overlapped Aluminium Tube-Tube Specimens under Constant Amplitude Multiaxial Loading

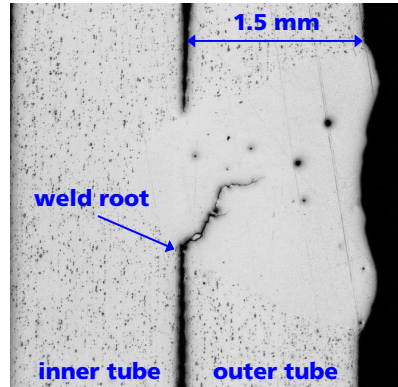


Experimental Results for Laserbeamwelded Magnesium Tubes



Typical failures

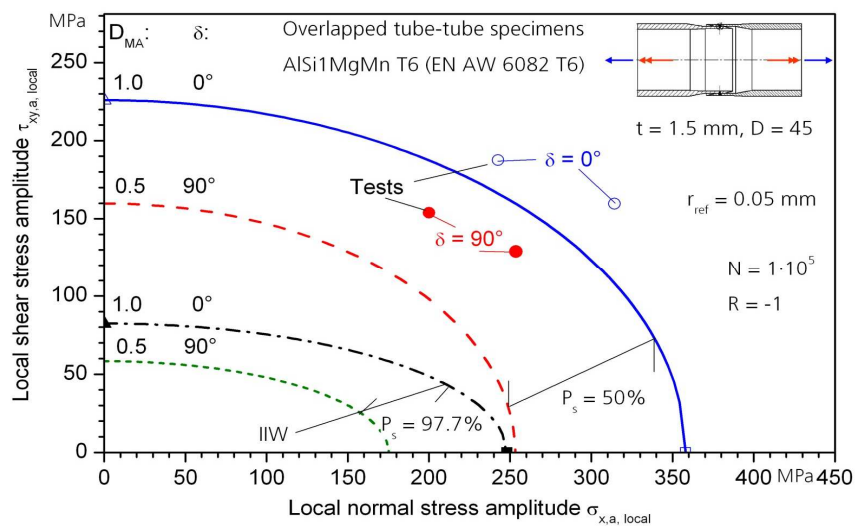
Independent of load combination and level!



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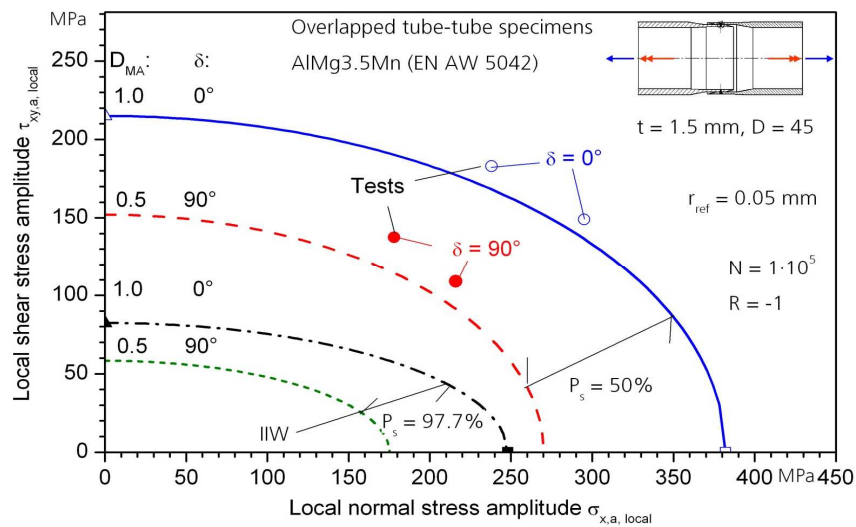
Assessment of Laserbeam-Welded Thin Aluminium Tube-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



D/A 8365e

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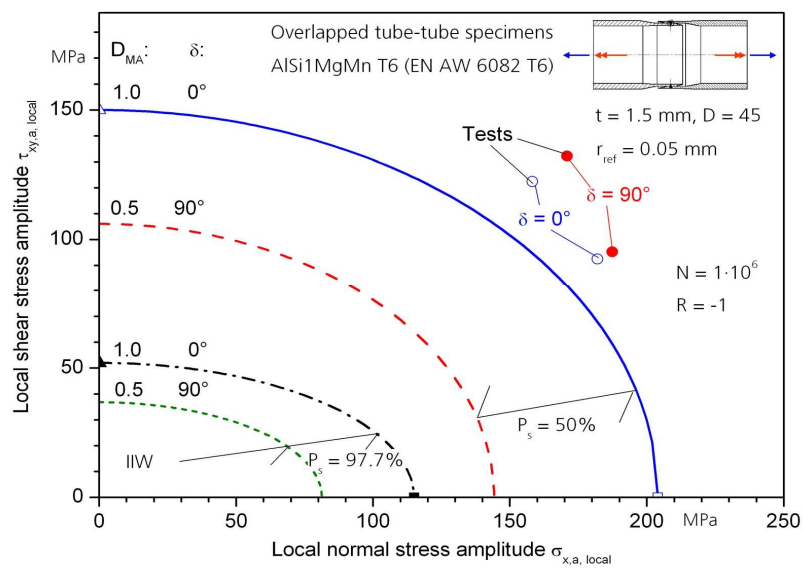
Assessment of Laserbeam-Welded Thin Aluminium Tube-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



DIA 8366e

Fraunhofer
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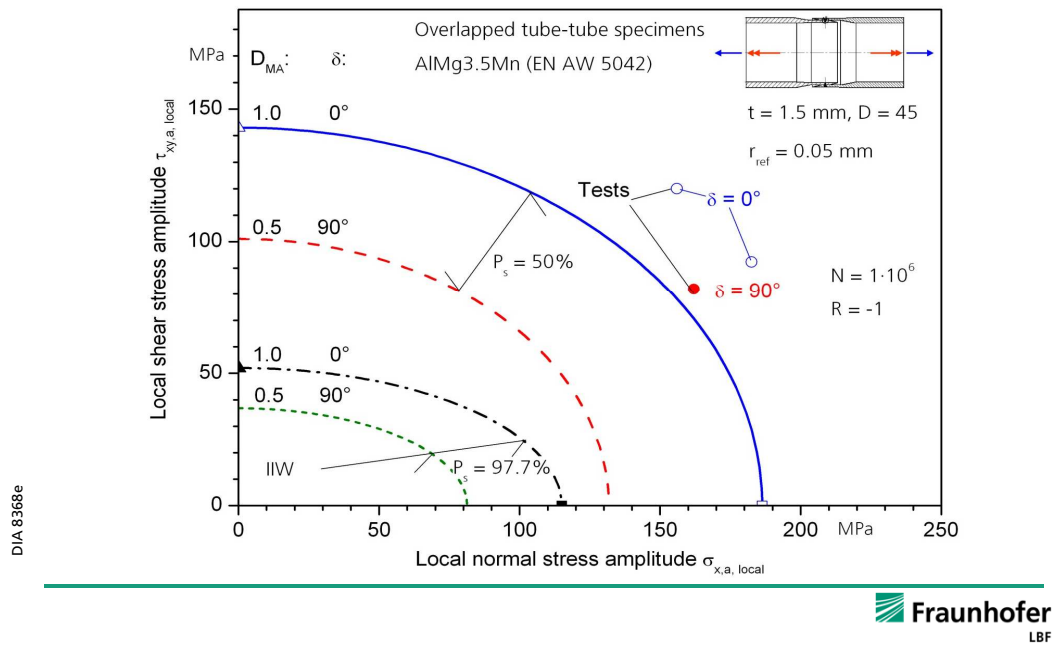
Assessment of Laserbeam-Welded Thin Aluminium Tube-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



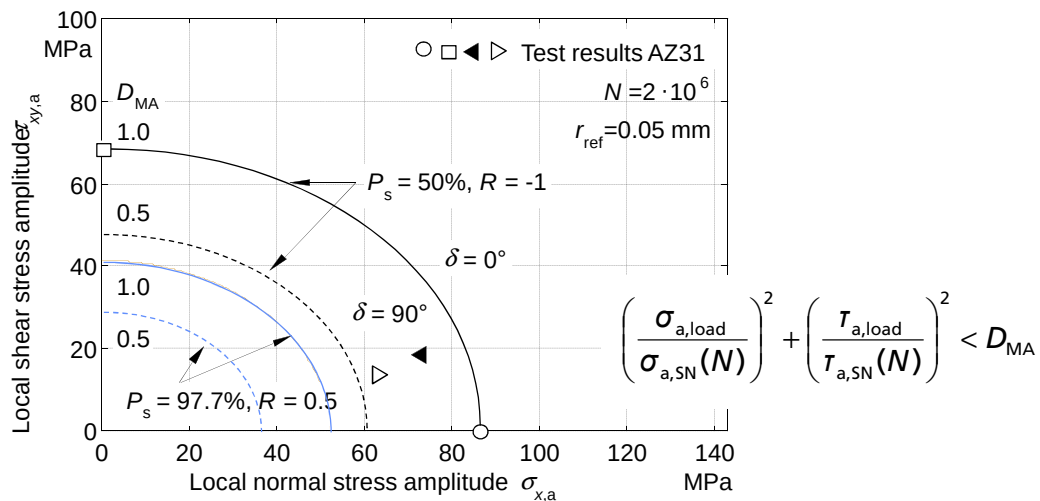
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Assessment of Laserbeam-Welded Thin Aluminium Tube-Tube Joints under In- and Out-of-Phase Multiaxial Loading (Local Stress System)



Evaluation of Test Results from Laserbeamwelded Magnesium Tubes According to Gough-Pollard Equation



Bewertung mehrachsiger Spannungszustände nach den IIV-Empfehlungen (2010)

$$\left(\frac{\Delta\sigma_{\text{Last}}}{\Delta\sigma_{\text{WL}}(N)} \right)^{k_\sigma} + \left(\frac{\Delta\tau_{\text{Last}}}{\Delta\tau_{\text{WL}}(N)} \right)^{k_\tau} \leq D_{\text{MA}}$$

$k_\sigma = k_\tau = 2$	Exponenten der Gough-Pollard Beziehung
$D_{\text{MA}} = 1.0$	Für konstante Hauptspannungsrichtungen, phasengleiche (proportionale) Belastung für Stahl (Eurocode 3, IIV) und Aluminium
$D_{\text{MA}} = 0.5$	Für sich ändernde Hauptspannungsrichtungen, phasenverschobene (nicht proportionale) Belastung für Stahl und duktiles Aluminium ^{*)}
$\Delta\sigma_{\text{Last}} = 2 \cdot \sigma_{\text{a,Last}}$ $\Delta\tau_{\text{Last}} = 2 \cdot \tau_{\text{a,Last}}$	Nenn-, Hot-Spot oder örtliche Spannungskomponenten des zweiachsigigen Spannungszustandes infolge der äußeren Belastung
$\Delta\sigma_{\text{WL}}(N) = 2 \cdot \sigma_{\text{a,WL}}(N)$ $\Delta\tau_{\text{WL}}(N) = 2 \cdot \tau_{\text{a,WL}}(N)$	Ertragbare Nenn-, Hot-Spot oder der örtliche Spannungen bei N Schwingspielen unter reiner Axialbelastung bzw. Biegung und reiner Torsion

^{*)} Für semi-duktilen Aluminiumlegierungen $D_{\text{MA}} = 1.0$

DIA 7898d

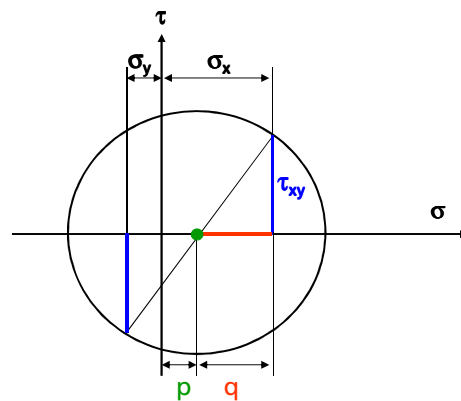
The Stress Space Curve Hypothesis SSCH

$$\sigma_{\text{eq}} = \sqrt{I_1^2 - (4 - \alpha) \cdot I_2}$$

$$\sigma_{\text{eq}} = \sqrt{\alpha \cdot p^2 + (4 - \alpha) \cdot (q^2 + \tau_{xy}^2)}$$

with

$$\begin{cases} p = (\sigma_x + \sigma_y) / 2 \\ q = (\sigma_x - \sigma_y) / 2 \\ \tau_{xy} \end{cases}$$

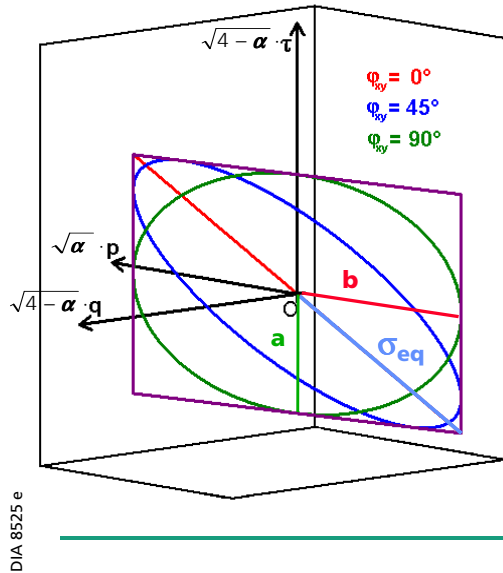


This approach can be linked to other hypotheses:

$$\begin{aligned} \alpha = 1 &\rightarrow \text{v. Mises} \\ \alpha = 0 &\rightarrow \text{Tresca} \end{aligned}$$

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Assessment of Multiaxial Loading



$$\sigma_{eq} = \sqrt{a^2 + b^2}$$

$$= \sqrt{\alpha \cdot p_a^2 + (4 - \alpha) \cdot (q_a^2 + \tau_a^2)}$$

a: semi minor axis of the ellipses
b: semi major axis of the ellipses

Consideration of out-of phase effects:

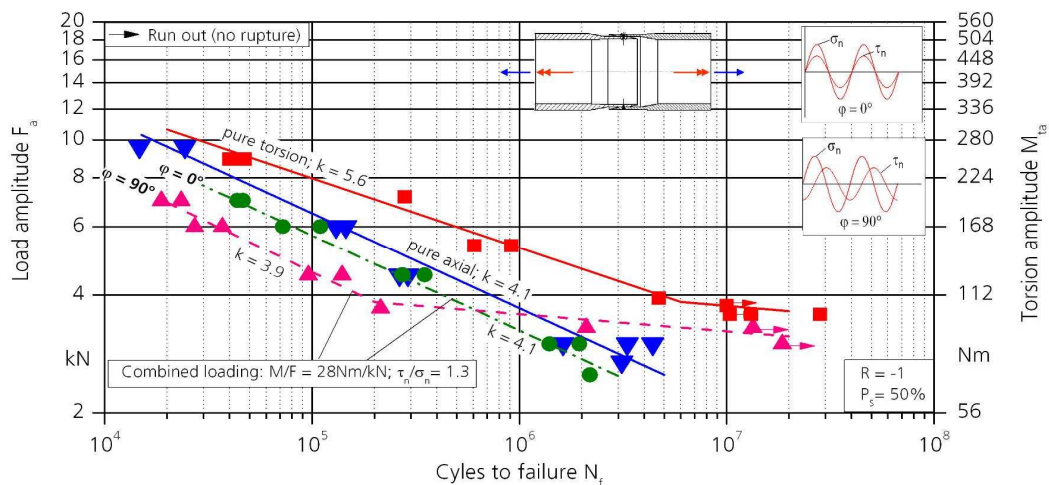
$$\sigma_{eq} = \sqrt{a^2 + b^2} \cdot \left(1 + \frac{\rho \cdot a \cdot b}{a^2 + b^2} \right)$$

Free parameter ρ :

$\rho > 0 \rightarrow$ lifetime decrease
 $\rho = 0 \rightarrow$ neutral behaviour
 $\rho < 0 \rightarrow$ lifetime increase

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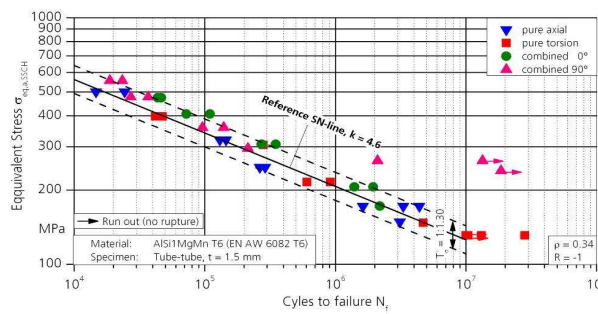
Experimental Results Constant Amplitude Loading



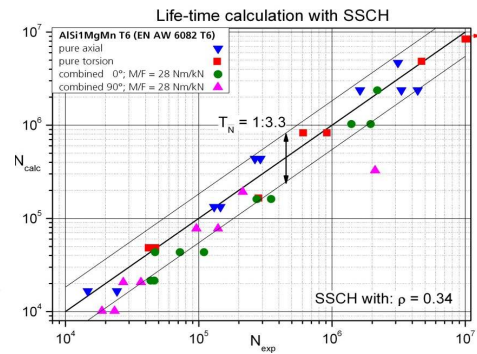
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Assessment of Multiaxial Loading with SSCH

$\sigma_{eq, SSCH}$ VS. N_{exp}



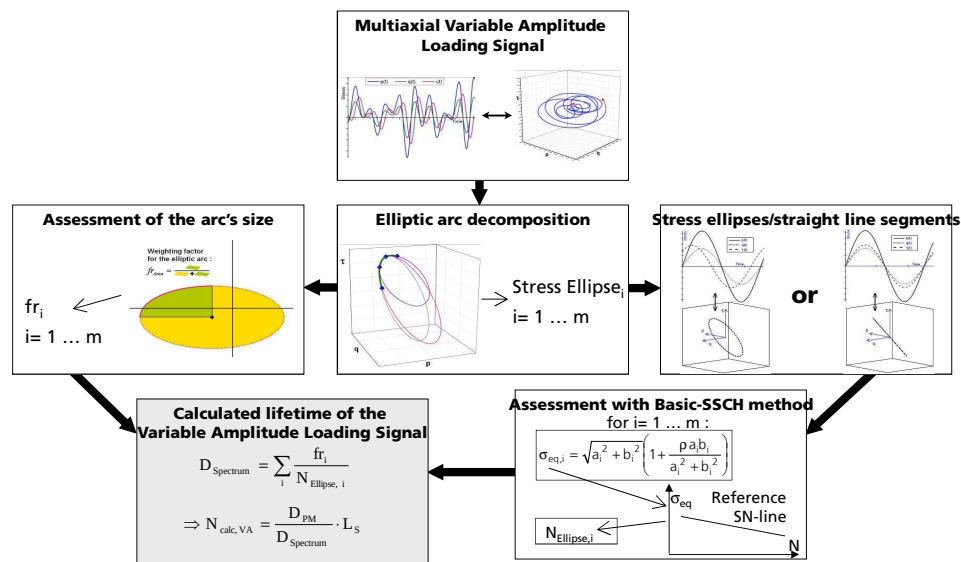
N_{calc} VS. N_{exp}



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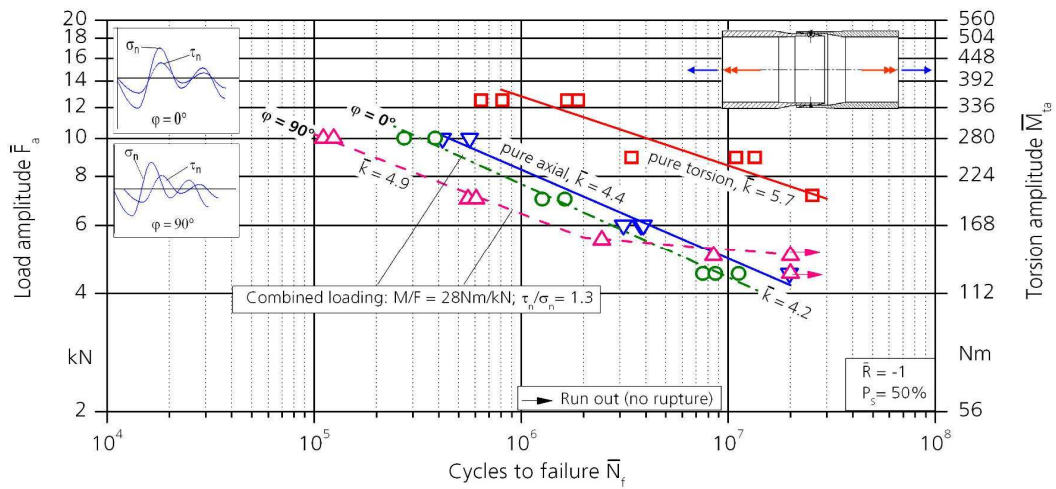
Assessment of Multiaxial Loading



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Experimental Results Variable Amplitude Loading

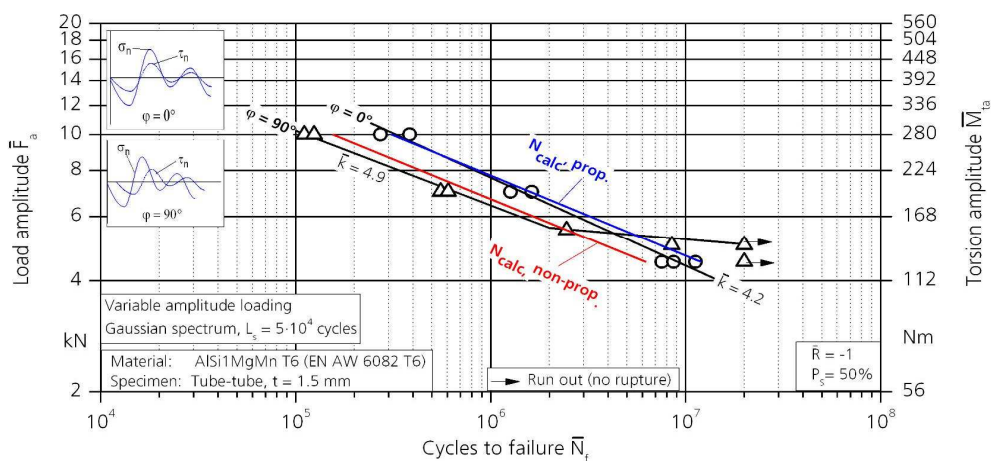


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Assessment of Multiaxial Loading

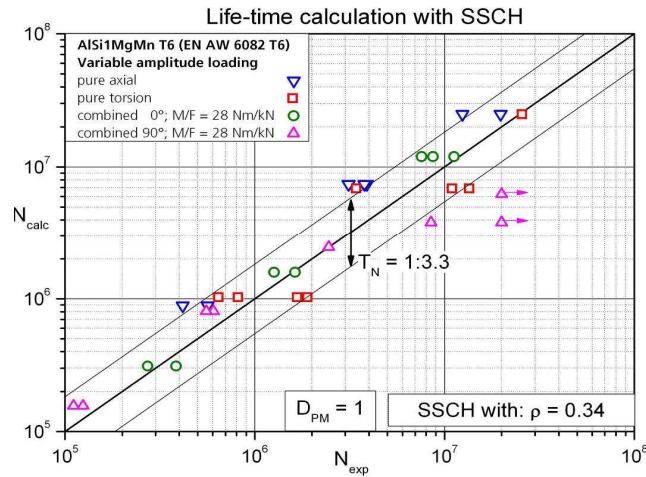
Calculated Lifetime N_{calc} for multiaxial spectrum loading



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N_{calc} versus N_{exp} for Variable Amplitude Loading



DIA 8528 e

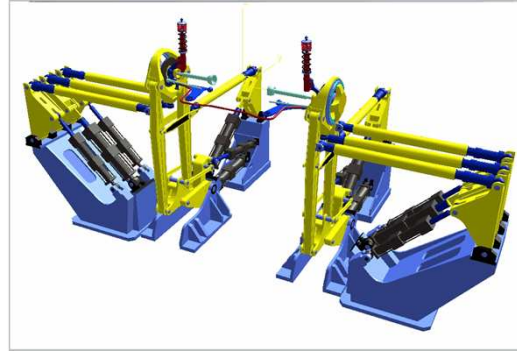
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Praktische Hinweise für die Auswahl einer geeigneten Hypothese

- Eine allgemeingültige Festigkeitshypothese ist nicht verfügbar.
- Für Werkstoffe mit geringer Bruchdehnung (A_5 etwa $< 5\%$) liefert die Normal- (Hauptspannungs-) Hypothese sowohl bei konstanten als auch drehenden Hauptspannungsrichtungen (infolge von Phasenverschiebungen), die die Lebensdauer verlängern, zufriedenstellende Ergebnisse (wenig duktiles / sprödes Werkstoffverhalten); maßgebend ist die Schnittebene mit der größten Normalspannung.
- Semiduktile Werkstoffe ($5\% < A_5 < 15\%$) zeigen ein neutrales Verhalten bezüglich der Drehung von Hauptspannungsrichtungen, keine Lebensdauerverlängerung oder –verkürzung. Die ungünstigste Kombination aus schnittebenen bezogenen Normal- und Schubspannungen ist für das Versagen verantwortlich. Die Gestaltänderungsenergiehypothese, als ob keine drehende Hauptspannungen (keine Phasenverschiebung) vorliegen würden, anwenden; dabei Vorzeichen (Druck oder Zug) beachten.
- Duktile Werkstoffe (A_5 etwa $> 15\%$) reagieren auf drehende Hauptspannungsrichtungen mit einer Verkürzung der Lebensdauer. Für das Versagen sind über verschiedene Schnittebenen interaktiv wirkende Schub- und Normalspannungen verantwortlich. Die Gestaltänderungsenergiehypothese, als ob keine drehende Hauptspannungen (keine Phasenverschiebung) vorliegen würden, anwenden; dabei Vorzeichen (Druck oder Zug) beachten. Allerdings die sich ergebende Vergleichsspannung pauschal um 20% erhöhen, um dem schädigenden Einfluß der drehenden Hauptspannungen auf die Lebensdauer Rechnung zu tragen.
- Die Grenzen von Hypothesen sind nur durch Einbeziehung von Labor- und Felderfahrungen feststellbar und so in die Berechnung einzubeziehen.

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12 Channel Road Simulator at LBF and Numerical MBS Model



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Design life: 300.000 km
Spectrum occurrence: $P_0 = 1\%$

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27-Kanal Ganzfahrzeugprüfstand

- ⑩ Iterationsgrößen sind Radlasten & lokale Messstellen
- ⑩ Lokale Messstellen zur Sicherstellung des gleichen Lastflusses im Prüfstand und Strecke
- ⑩ Lokale Messstellen zur Bewertung der Simulationsgüte
- ⑩ Lokale Messstellen kalibriert in kN oder in kritischen Bereichen in MPa



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Vorlesung bei der EPFL, Lausanne, 9. September 2015

Thanks for the kind invitation
and for your kind attention!